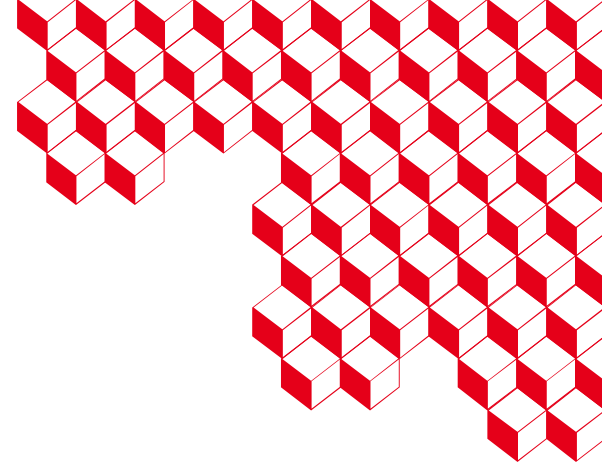




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## **An approach to thermomechanical modelling of welding and additive manufacturing in Cast3M**

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# Introduction



- State of the art
  - Numerical **welding simulation** uses **finite element codes** for structural analysis
  - It takes benefit of decades of experience in thermomechanical modelling that have led to:
    - Robust and proven numerical methods
    - Performant nonlinear solvers
    - Numerous mechanical behavior laws
    - Accessible and validated material data
- However, modelling **welding and additive manufacturing** processes presents **two specific characteristics**:
  - Mobile heat input, also variable in time
  - Material deposition

Subject of today's talk!



# Outlines

## 1. Mobile heat input

Heat source models

Spatiotemporal evolution

## 2. Material deposition

Model geometry evolution

Energy conservation

## 3. Applications

# Mobile heat input

- Heat source models: [why?](#)

- Heat input processes impose heat fluxes (electric arc, laser...)

However:

- Thermal modelling of these processes uses [heat source models](#) to account for [hydrodynamic effects](#) in the molten pool

- Example: Goldak<sup>1</sup> heat source model

Good representation of the molten pool but 8 parameters to identify (at least)

- Heat source models [in Cast3M](#)

- Development of heat source models:

MOD1 = MODE GEO1 THERMIQUE SOURCE... ;

- Two gaussian heat source models:

- ... SOURCE GAUSSIENNE ELLIPTIQUE ;

- ... SOURCE GAUSSIENNE ELARGIE ;

Goldak<sup>1</sup> heat source model

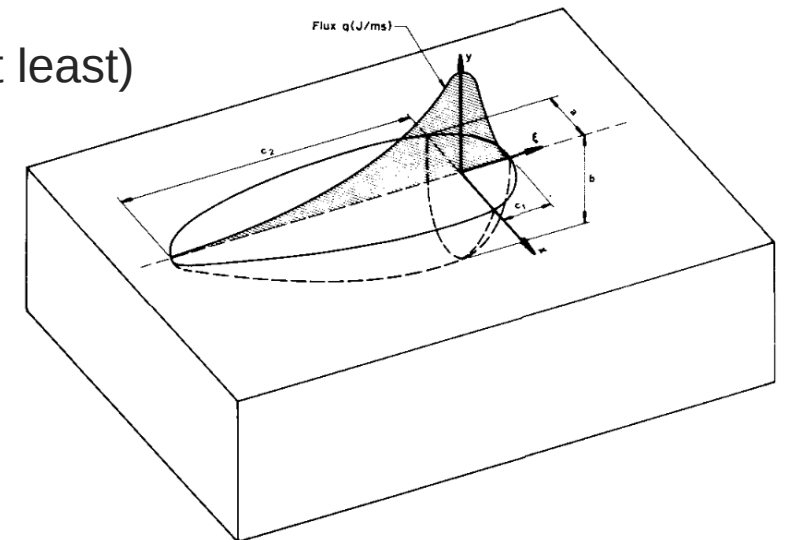


Fig. 3—Double ellipsoid heat source configuration together with the power distribution function along the  $\xi$  axis.

<sup>1</sup>J. Goldak, A. Chakravarti, M. Bibby. "A new finite element model for welding heat sources", Metallurgical Transactions B, vol. 15, 299–305, 1984.

# Mobile heat input

■ “SOURCE GAUSSIENNE ELLIPTIQUE”

■ “Basic” gaussian heat source model

■ Thermal power density distribution function:

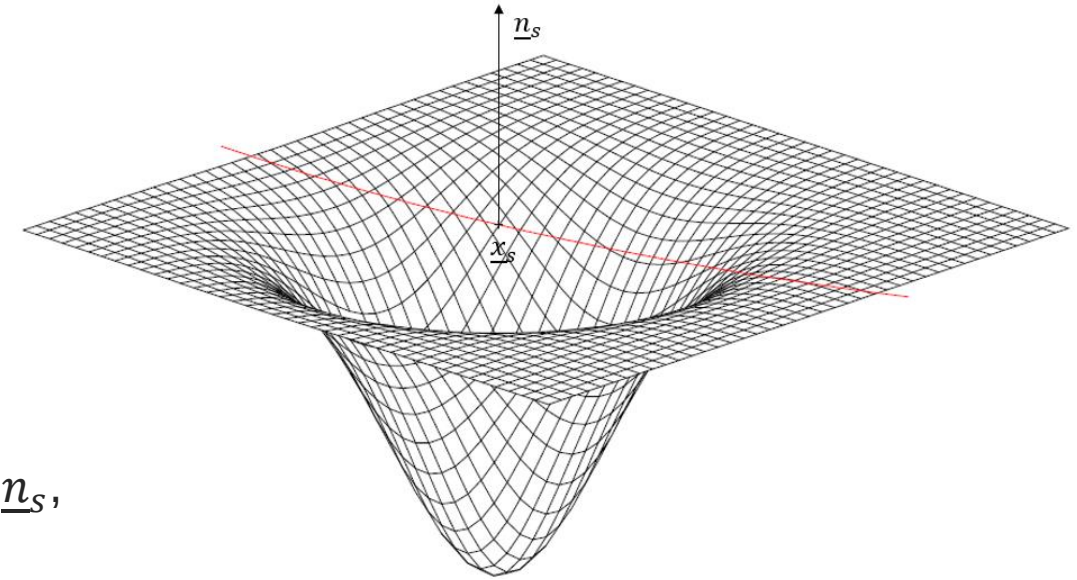
$$s(\underline{x}) = s_0 e^{-2((\tilde{r}/r_g)^2 + (\tilde{z}/z_g)^2)}$$

$(\tilde{r}, \tilde{z})$  space coordinates in local cylindrical frame of axis  $\underline{n}_s$ ,  
and  $s_0$  such that:

$$\iiint_{\tilde{z} < 0} s(\underline{x}) dV = Q_s$$

■ 5 parameters:

- $\underline{x}_s, \underline{n}_s$  : location and orientation
- $r_g, z_g$  : shape
- $Q_s$  : power



Expression of  $s_0$  for 3 calculation modes: plane, axisymmetric and tridimensional.

| mode de calcul                                                                                                                                                                       | 2D plan                             | 2D axisymétrique                                                          | tridimensionnel                                  |
|--------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------|-------------------------------------|---------------------------------------------------------------------------|--------------------------------------------------|
| expression de $s_0$                                                                                                                                                                  | $\frac{4}{\pi} \frac{Q_s}{r_g z_g}$ | $\sqrt{\frac{2}{\pi^3}} \frac{Q_s}{z_g (\kappa_1 + \kappa_2 + \kappa_3)}$ | $\sqrt{\frac{2^5}{\pi^3}} \frac{Q_s}{r_g^2 z_g}$ |
| avec, pour le mode axisymétrique :                                                                                                                                                   |                                     |                                                                           |                                                  |
| $\kappa_1 = \frac{r_g^2}{4} e^{-2(r_s/r_g)^2}, \quad \kappa_2 = \sqrt{\frac{\pi}{8}} r_g r_s, \quad \kappa_3 = \sqrt{\frac{\pi}{8}} r_g r_s \operatorname{erf}(\sqrt{2} r_s / r_g),$ |                                     |                                                                           |                                                  |
| $r_s$ étant la 1 <sup>re</sup> coordonnée du centre de la source dans le mode axisymétrique.                                                                                         |                                     |                                                                           |                                                  |

# Mobile heat input

## ■ “SOURCE GAUSSIENNE ELARGIE”

- Simplified heat source model to represent **sweep welding**

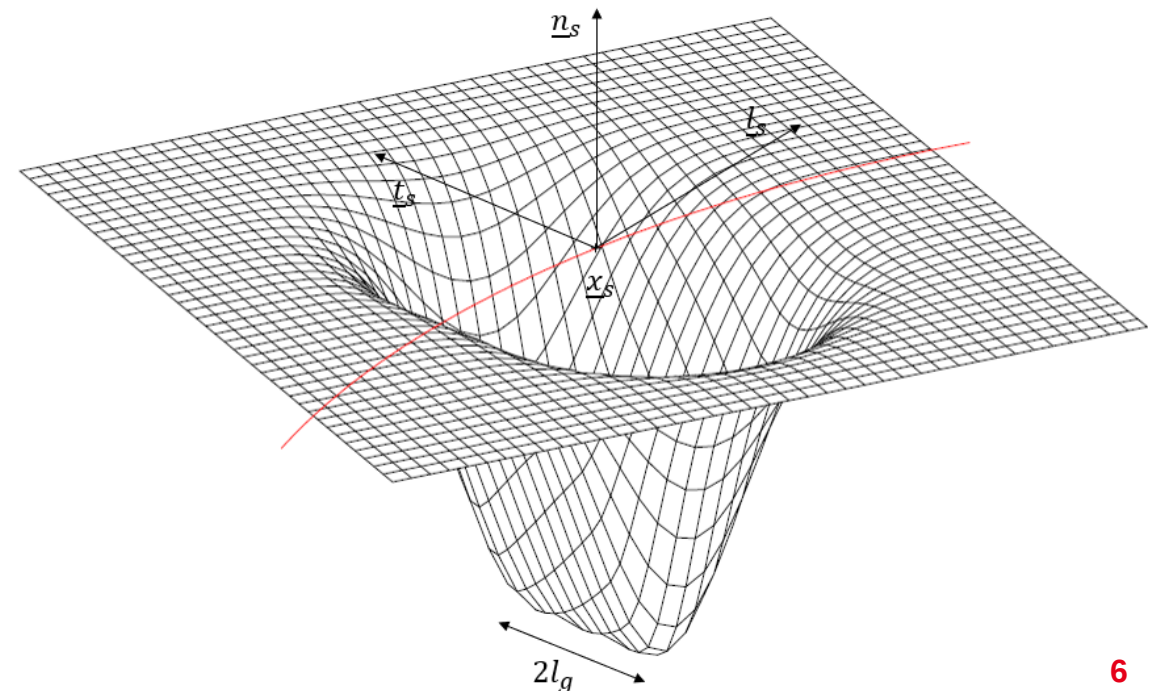
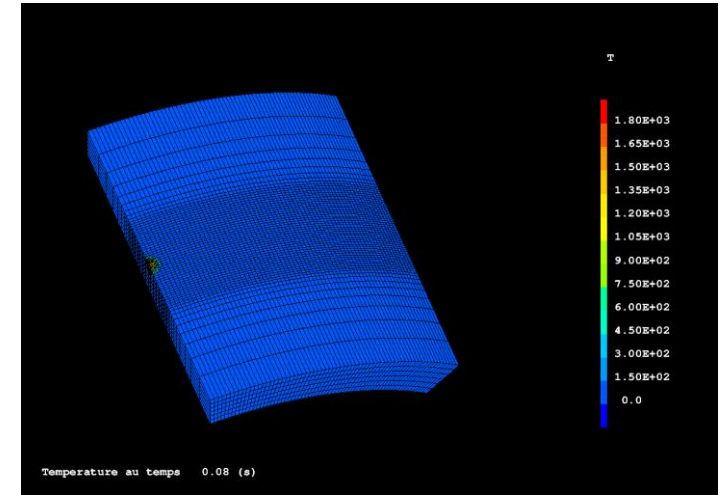
- Thermal power density distribution function (only in 3D mode):

$$s(\underline{x}) = \frac{Q_s}{(\sqrt{\pi^3/32})r_g^2 z_g + (\pi/2)r_g z_g l_g} e^{-2((u/r_g)^2 + (\max(0, |v| - l_g)/l_g)^2 + (w/z_g)^2)}$$

$(u, v, w)$  space coordinates in local frame  $(\underline{l}_s, \underline{t}_s, \underline{n}_s)$

- 7 parameters:

- $\underline{x}_s, \underline{t}_s, \underline{n}_s$  : location and orientation
- $r_g, z_g, l_g$  : shape
- $Q_s$  : power



# Mobile heat input

## ■ Mobile heat source modelling

### ■ Based on:

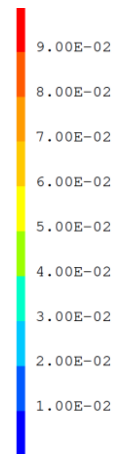
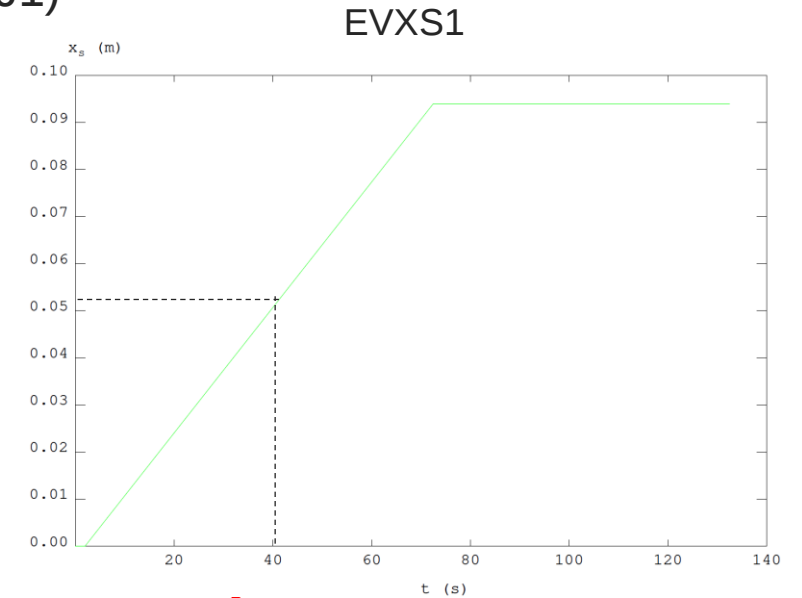
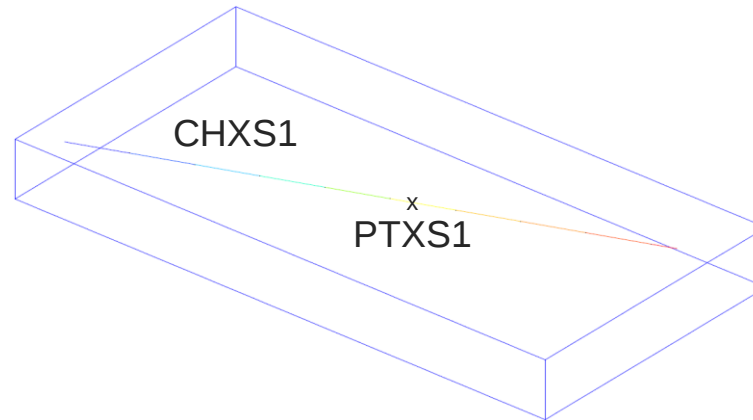
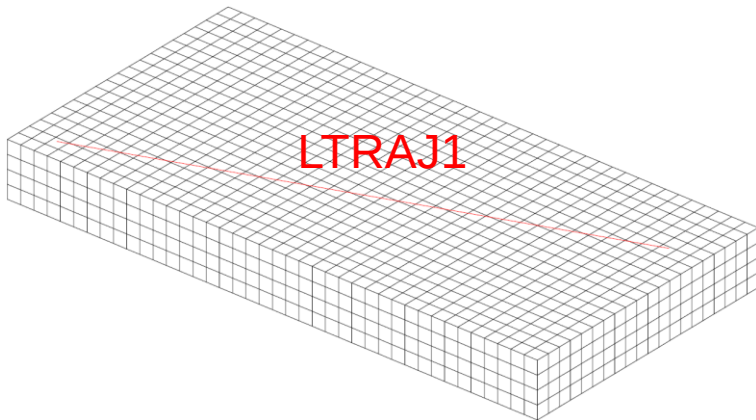
- Heat source trajectory, modelled as an oriented meshed line (LTRAJ1)
- Time evolution of its displacement along this line (EVXS1)

### ■ In Cast3M, definition of a CHARGEMENT type object:

```
CHXS1 = LTRAJ1 COOR CURV ;
CGXS1 = CHAR TRAJ CHXS1 EVXS1 ;
```

### ■ Use for a given time TPS1:

```
PTXS1 = TIRE CGXS1 TRAJ TPS1 ;
```



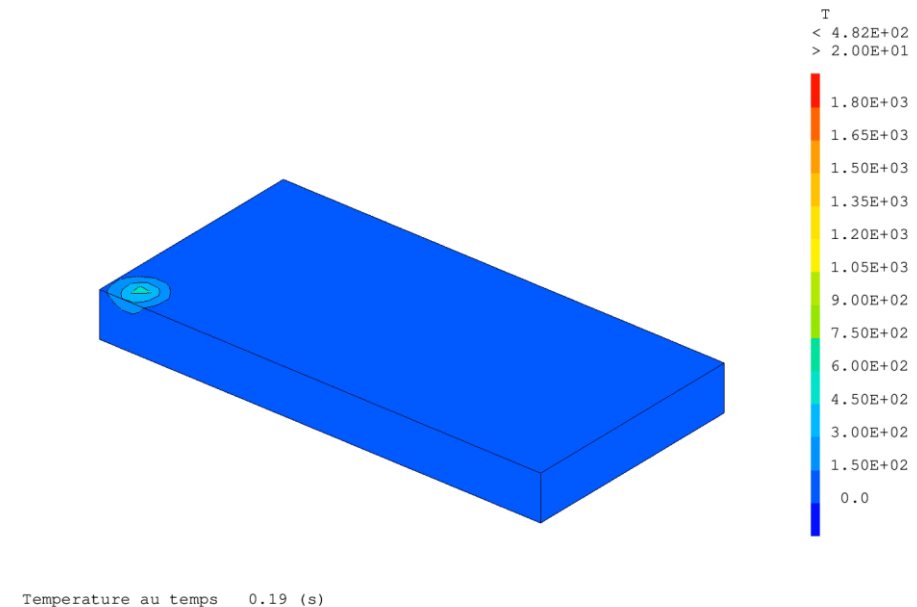
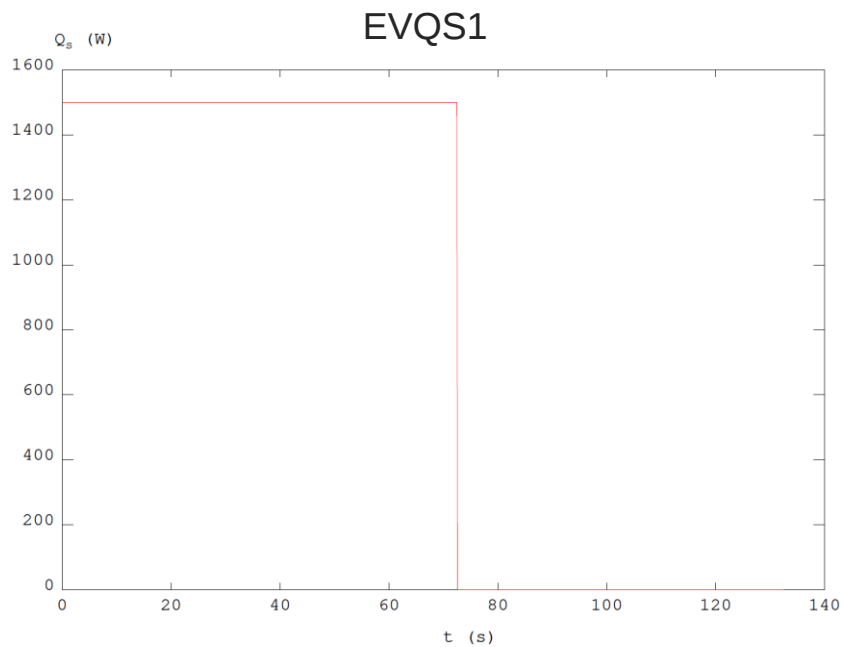
# Mobile heat input

## ■ Mobile heat source modelling

### ■ Connection to the heat source models in Cast3M:

```
MODQS1 = MODE GEO1 THERMIQUE SOURCE GAUSSIENNE... ;
MATQS1 = MATE MODQS1 ORIG CGXS1 QTOT EVQS1 RGAU RG1 ZGAU ZG1 DIRE NS1... ;
```

- the location of the heat source can be updated at any time by giving CGXS1, as well as the heat source intensity by giving EVQS1





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# Material deposition

## ■ Geometry evolution

- Basic idea: **add blocks of finite elements** du simulate material deposition



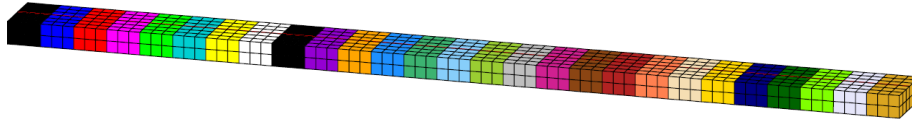
## ■ In practice:

Define **as many mesh as there are geometries** involved in the simulation

At time  $t$ , **update the geometry** according to the following rule:



⋮



$t_0 \leq t < t_1$  geometry GEO1

$t_1 \leq t < t_2$  geometry GEO2

⋮

$t_{n-1} \leq t < t_n$  geometry GEO $n$

- Data structure in Cast3M: create a **list of meshes** (LISTOBJE) and a timetable (LISTREEL):

LGE01 = **ENUM** GEO1 GEO2 GEO3 ... GEO $n$  ;

LTPS1 = **PROG** t0 t1 t2 ... tn ;



# Material deposition

## ■ Model evolution

### ■ Define as many models as there are meshes in LGEO1:

```
LMOD1  = ENUM ;
LMAT1  = ENUM ;
REPE B1 (DIME LGEO1) ;
    GEOI1  = LGEO1 EXTR &B1 ;
    MODI1  = MODE GEOI1 MECANIQUE ELASTIQUE ... ;
    MATI1  = MATE MODI1 YOUN YM1 NU NU1 ... ;
    LMOD1  = LMOD1 ET MODI1 ;
    LMAT1  = LMAT1 ET MATI1
FIN B1 ;
CGMOD1  = CHAR MODE LTPS1 LMOD1 ;
CGMAT1  = CHAR MATE LTPS1 LMAT1 ;
```

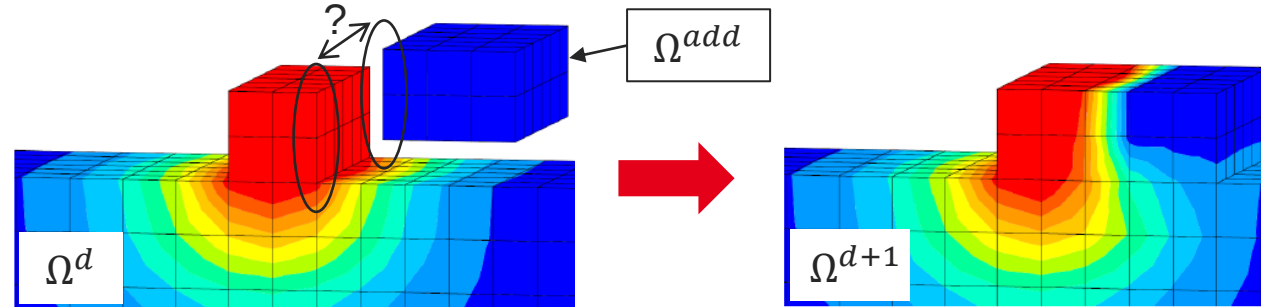
### ■ Use in PASAPAS input table:

```
TAB1.MODELE          = CGMOD1 ;
TAB1.CARACTERISTIQUES = CGMAT1 ;
```

During resolution, **PASAPAS updates the model and its characteristics**, which makes the geometry evolving.

# Material deposition

- Thermal analysis: **temperature initialization** during model change



- Temperature at interface  $\Omega^d \cap \Omega^{add}$  ? → **energy conservation**

$$C^{d+1} T^{d+1} = C^d \cdot T^d + C^{add} \cdot T^{add},$$

where  $C^x$  are the heat capacity matrices and  $T^x$  the nodal temperatures.

- With:

$$C^{d+1} = C^d + C^{add},$$

we get:

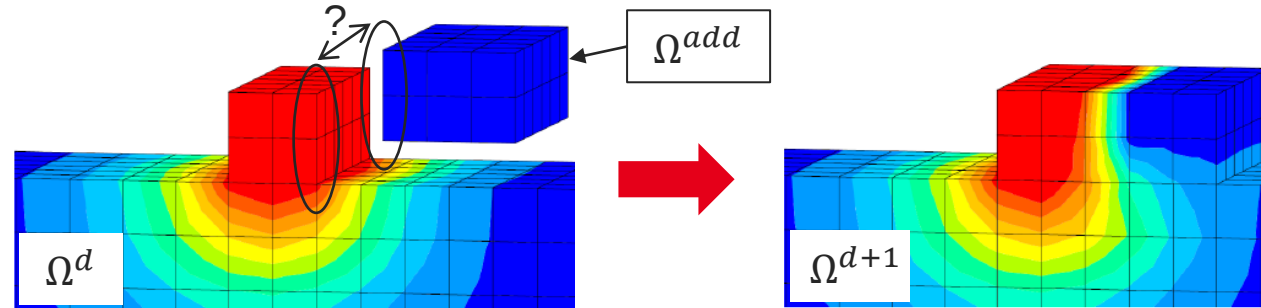
$$T^{d+1} = (C^d + C^{add})^{-1} (C^d \cdot T^d + C^{add} \cdot T^{add}).$$

Important remark:  $T^{d+1}$  is temperature **at the beginning of the next time step.**



# Material deposition

- Mechanical analysis: **displacements and stresses initialization** during model change



- Static analysis: **no mechanical energy in  $\Omega^{add}$**

- Displacement field at interface  $\Omega^d \cap \Omega^{add}$ :

$$u(\Omega^d \cap \Omega^{add}) = u^d \Rightarrow \varepsilon^{add} \neq 0 \Rightarrow E_m^{\varepsilon, add} \neq 0$$

- Moreover:

$$\varepsilon^{th, add} = \varepsilon^{th}(T^{d+1}) \neq 0 \Rightarrow E_m^{th, add} \neq 0$$

- **Initial stress free strain  $\varepsilon^L$  applied to the added part** to cancel this input of mechanical energy:

$$\varepsilon^L = \varepsilon^{add} - \varepsilon^{th}(T^{d+1}) \Rightarrow \sigma^{add} = L^e : (\varepsilon^{add} - \varepsilon^{th} - \varepsilon^L) = 0$$



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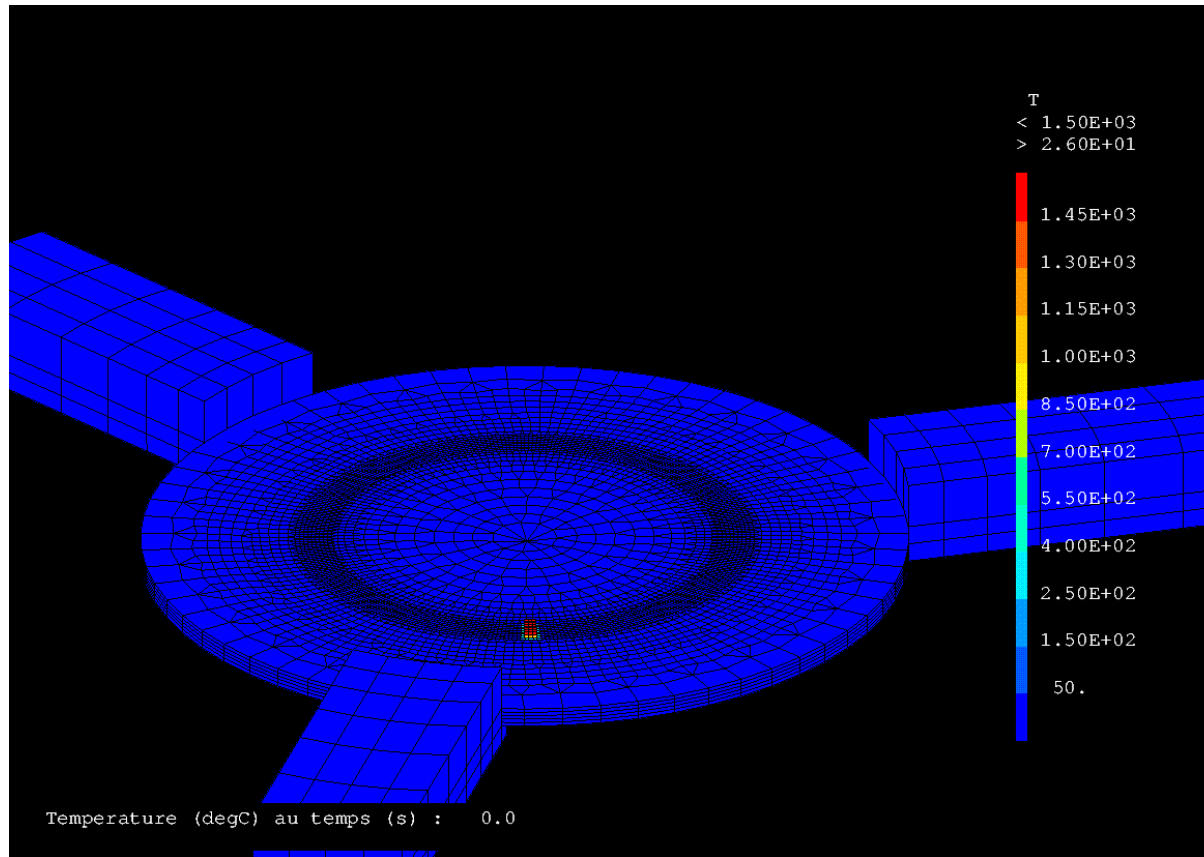
Energy conservation

## 3. Applications

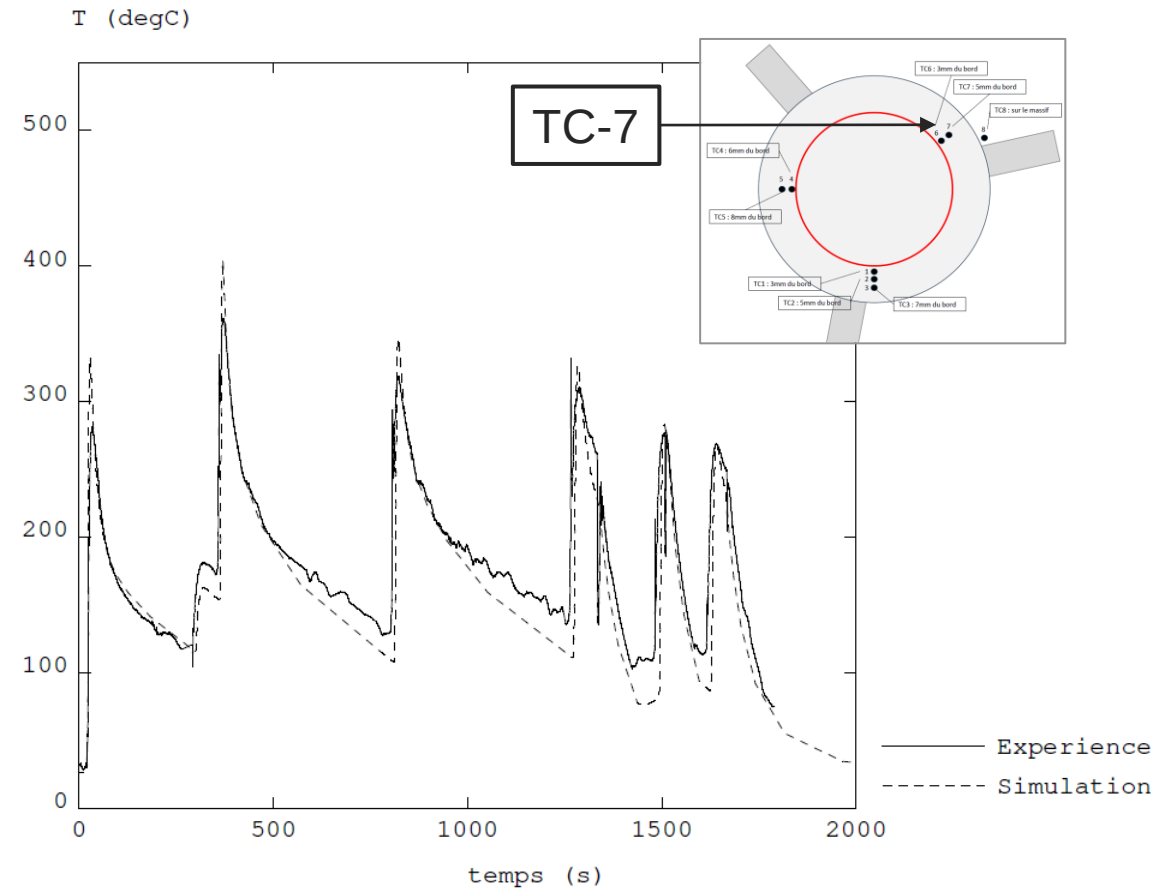
# Applications

- Example of **thermal analysis** : tube manufacturing with WAAM process <sup>1</sup>

Time evolution of temperature node field (°C)



Time evolution of temperature (°C) at a thermocouple location

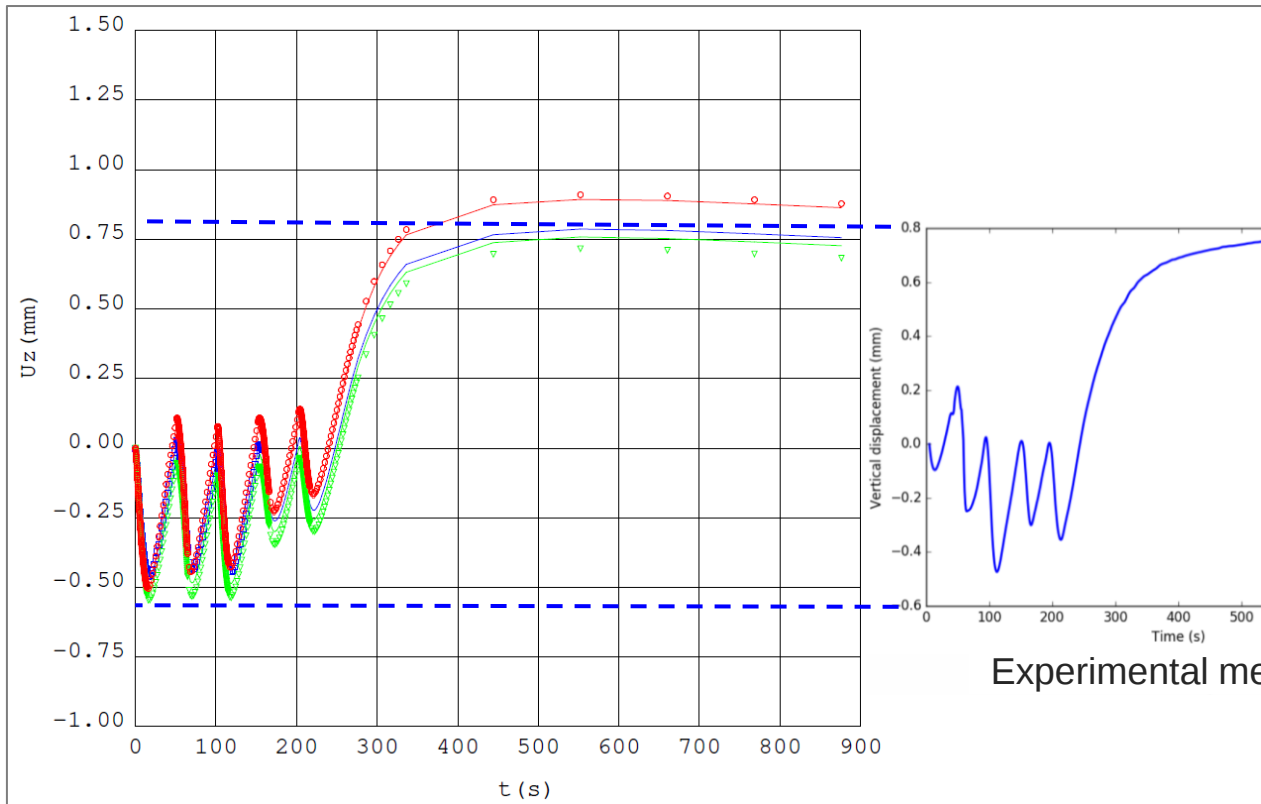


<sup>1</sup>S. Pascal, M. Garnier, M. Garric, J. Le Mercier. *Modélisation thermomécanique de la fabrication additive arc-fil d'un tube droit avec Cast3M*, 16ieme Colloque National en Calcul des Structures (CSMA 2024), CNRS, CSMA, ENS Paris-Saclay, CentraleSupélec, mai 2024, Hyères, France.

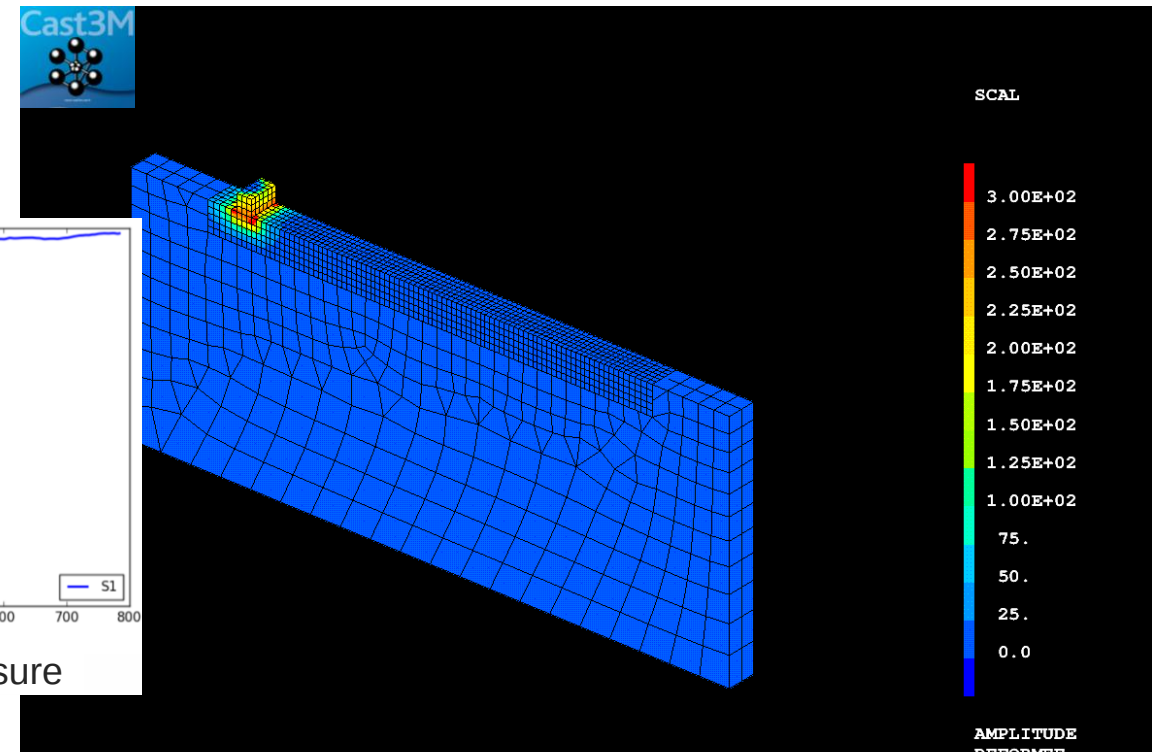
# Applications

- Example of **mechanical analysis** : wall manufacturing with WAAM process<sup>1,2</sup>

Time evolution of  $U_z$  displacement (mm) at the plate end



Time evolution of von Mises stress field (MPa)



Linear isotropic hardening, large displacement analysis

<sup>1</sup>C. Cambon, S. Rouquette, I. Bendaoud, C. Bordreuil, R. Wimpory, F. Soulié. *Thermo-mechanical simulation of overlaid layers made with Wire + Arc Additive Manufacturing and GMAW-Cold Metal Transfer*, Welding in the World, Vol. 64 No. 8, pp 1427-1435.

<sup>2</sup>S. Pascal, S. Gounand. *Modélisation thermomécanique d'un cas d'étude de fabrication additive arc-fil*, 15<sup>ieme</sup> Colloque National en Calcul des Structures (CSMA 2022), Université Polytechnique Hauts-de-France, [UPHF], mai 2022, Hyères, France.

# Applications

- Example of **mechanical analysis** : wall manufacturing with WAAM process<sup>1</sup>

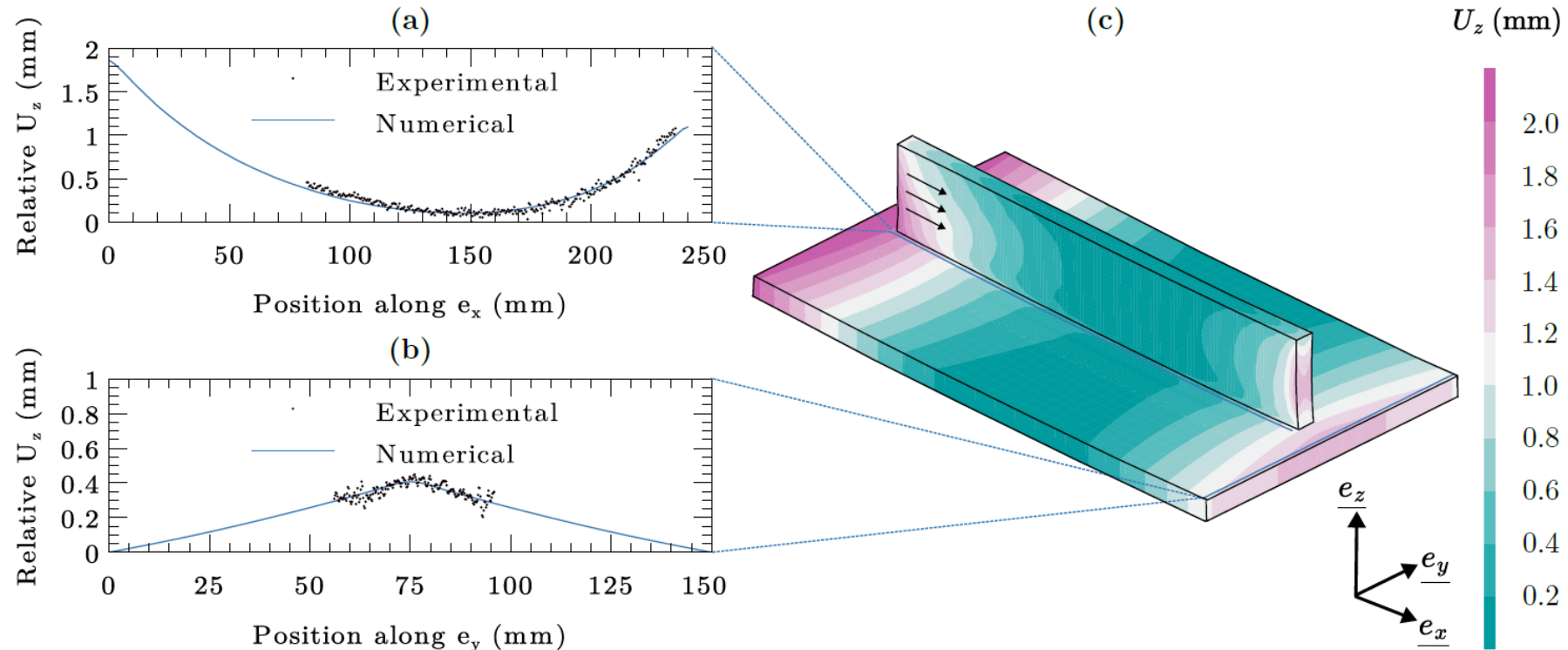
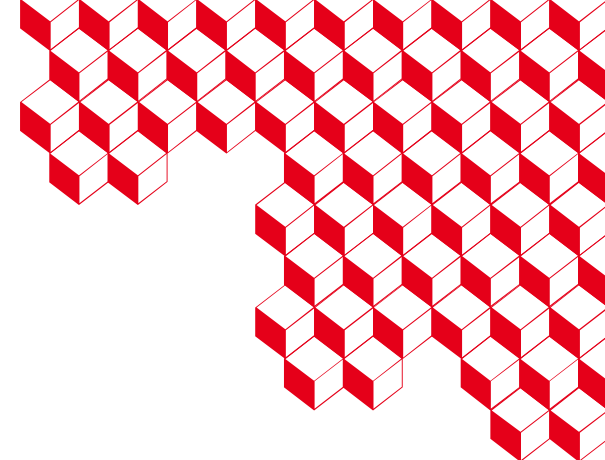


Figure 10. Comparison of the experimental and numerical displacements in the  $\underline{e}_z$  direction at the end of the deposition after cooling and unclamping the WAAM specimen: (a) experimental laser profilometry and numerical  $\underline{e}_z$ -displacement evolution along the  $\underline{e}_x$  axis near the manufacturing area; (b) experimental laser profilometry and numerical  $\underline{e}_z$ -displacements evolution along the  $\underline{e}_y$  axis at the extremity of the substrate; (c) numerical estimation of the  $\underline{e}_z$ -displacements after complete cooling and unclamping of the component.

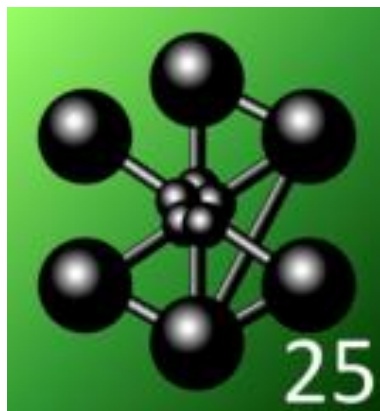
<sup>1</sup>D. Artieres. *Microstructure et propriétés de l'acier 316L élaboré par fabrication additive par dépôt de fil (WAAM & WLAM) : caractérisation expérimentale et modélisation*, thèse de doctorat, Université Paris sciences et lettres, 2025.



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# Merci de votre attention



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