

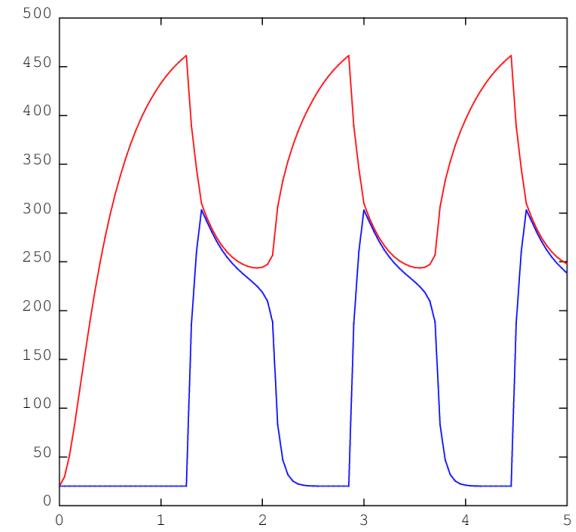
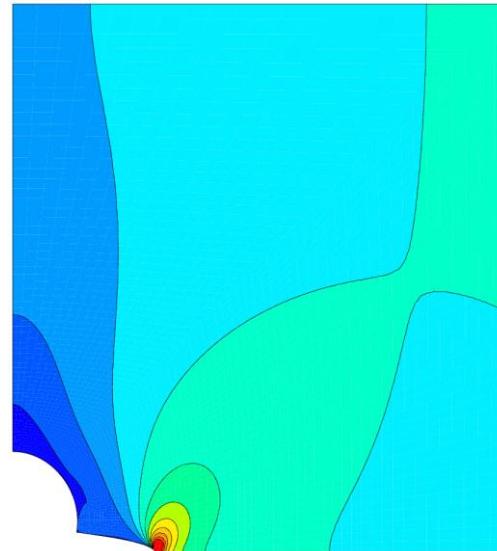
DE LA RECHERCHE À L'INDUSTRIE



www.cea.fr

THE PASAPAS PROCEDURE AND THE USERS PROCEDURES

AVAILABLE ON : <HTTP://WWW-CAST3M.CEA.FR/INDEX.PHP?XML=FORMATIONS>



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SUMMARY

- Introduction to Cast3M and reminders on **PASAPAS**
- **PASAPAS** operation
- Finding the mechanical equilibrium: **UNPAS**
 - Exercise 1: following load
 - Exercise 2: failure by elements removal
- Finding the thermal equilibrium: **TRANSNON**
 - Exercise 3: variable heat source
 - Exercise 4: thermo-mechanical contact

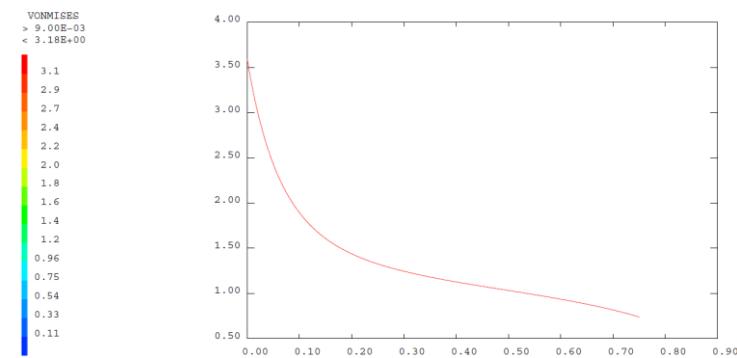
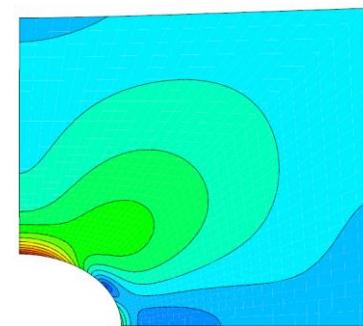
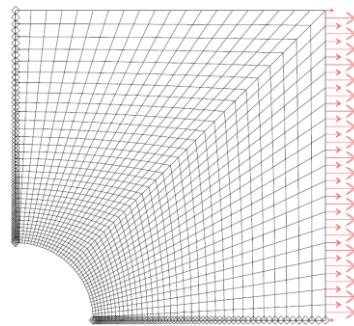
INTRODUCTION TO CAST3M

REMINDERS ON PASAPAS

CAST3M, QUID ?

A **finite element** software
for **structural** and **fluid mechanics**

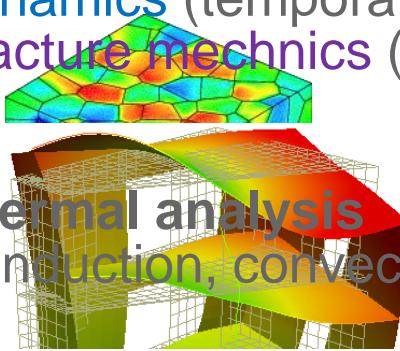
- Partial differential equations solved with the finite element method
- Complete software: solver, pre-processing and post-processing, visualization



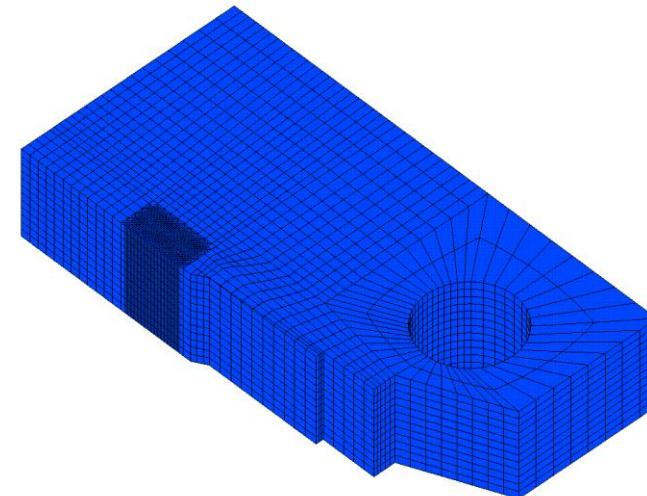
- Based on an objet-oriented programming language: **Gibiane**

MODELING IN NUMEROUS AREAS

- **Structural mechanics (historical field)**
Quasi-static (non linear materials, geometry, boundary conditions)
Contact/friction, **Buckling**
Dynamics (temporal, modal, fluid structure interaction)
Fracture mechanics (XFEM, dynamic propagation, cohesive zone model)



- **Thermal analysis**
Conduction, convection, ra



SCAL



- **Fluid mechanics**

- **Metallurgy**

- **Magnetostatics**

- **Multi species diffusion (F**

- **Thermo-hydro-mechanical**

Temperatu

Proportions de BAIN au temps : 0.00000E+00

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CAST3M FECIT

THE PASAPAS PROCEDURE

■ Objective

incremental solving of non linear **progressive thermal** and **mechanical** problems

time can be physical (e.g. thermal transients)
or not (e.g. plasticity with progressive loading)

→ *time or pseudo-time is called the evolution parameter*

■ Non linear phenomena considered

behavior (plasticity, damage, variable material properties, ...)

geometry (large displacements)

strains (large rotations)

boundary conditions (radiation, friction, following pressure, ...)

PASAPAS USE

■ Create a table containing all the data:

TAB1	= TABL ;	
TAB1 . MODELE	= MOD1 ET MOD2 ;	<i>some indexes are required</i>
TAB1 . CARACTERISTIQUES	= MAT1 ET MAT2 ;	<i>other are optional</i>
TAB1 . BLOCAGES_MECANIQUES	= BL01 ;	
TAB1 . CHARGEMENT	= CHA1 ET CHA2 ET CHA3 ;	
TAB1 . TEMPS_CALCULES	= PROG 0.1 'PAS' 0.1 50. ;	
TAB1 . TEMPS_SAUVES	= PROG 4. 8. 15. 16. 23. 42. ;	
TAB1 . PRECISION	= 1.E-6 ;	
TAB1 . GRANDS_DEPLACEMENTS	= VRAI ;	
...		

■ Procedure call:

PASAPAS TAB1 ;

■ Results post-processing

OVERVIEW OF INPUT PARAMETERS

■ General

MODELE (MMODEL)

CARACTERISTIQUES (MCHAML)

CHARGEMENT (CHARGEUME)

*Equations to solve, F. E. formulation (**MODE**)*

*Material and/or geometrical parameters (**MATE**)*

*Boundary conditions and loading variation during calculation (**CHAR**)*

■ Thermal analysis

BLOCAGES_THERMIQUES (RIGIDITE)

CELSIUS (LOGIQUE)

TEMPERATURES . 0 (CHPOINT)

*Stiffness matrix associated to the imposed values of the unknowns (DIRICHLET boundary conditions) (**BLOQ**)*

=VRAI (true) if temperature unit is CELSIUS

Initial conditions

■ Mechanics

BLOCAGES_MECANIQUES (RIGIDITE)

GRANDS_DEPLACEMENTS (LOGIQUE)

DEPLACEMENTS . 0 (CHPOINT)

CONTRAINTES . 0 (MCHAML)

VARIABLES_INTERNES . 0 (MCHAML)

DEFORMATIONS_INELASTIQUES . 0 (MCHAML)

*Stiffness matrix associated to the imposed values of the unknowns (DIRICHLET boundary conditions) (**BLOQ**)*

Equilibrium checked on the deformed mesh

Initial conditions

OVERVIEW OF INPUT PARAMETERS

■ Mechanics (dynamics)

DYNAMIQUE (LOGIQUE)

AMORTISSEMENT (RIGIDITE)

VITESSES . 0 (CHPOINT)

ACCELERATIONS . 0 (CHPOINT)

=VRAI (true) *for dynamics calculations*

Damping matrix

Initial conditions

{}

■ List of the values of evolution parameter

TEMPS_CALCULES (LISTREEL)

List of the values of the evolution parameter (or time) for which results are requested

TEMPS_SAUVES (LISTREEL)

List of times for which results are saved

OPTI 'SAUV' 'my_file' ;

TEMPS_SAUVEGARDES (LISTREEL)

List of times for which SAUV operator is called

MES_SAUVEGARDES (TABLE)

Results to be stored in addition to those automatically stored (total strains, ...)

OVERVIEW OF OUTPUT PARAMETERS

■ Results are stored in the table

TEMPS (TABLE)

Times values, corresponding to « TEMPS_SAUVES »

TEMPERATURES (TABLE)

PROPORTIONS_PHASE (TABLE)

DEPLACEMENTS (TABLE)

CONTRAINTES (TABLE)

DEFORMATIONS_INELASTIQUES (TABLE)

VARIABLES_INTERNES (TABLE)

REACTIONS (TABLE)

VITESSES (TABLE)

ACCELERATIONS (TABLE)

*Fields (solution) calculated for each stored time
« TEMPS_SAUVES »*

POST PROCESSING (EXAMPLES)

- **Solution fields extraction:**
from the table index

SIG1 = TAB1 . CONTRAINTES . 5 ;

or from the calculation time

SIG1 = PECHE TAB1 'CONTRAINTES' 28.3 ;

- **Graphical mode, interactive plot (limited):**
EXPLORER TAB1 ;

- **Evolution of calculated field with time:**

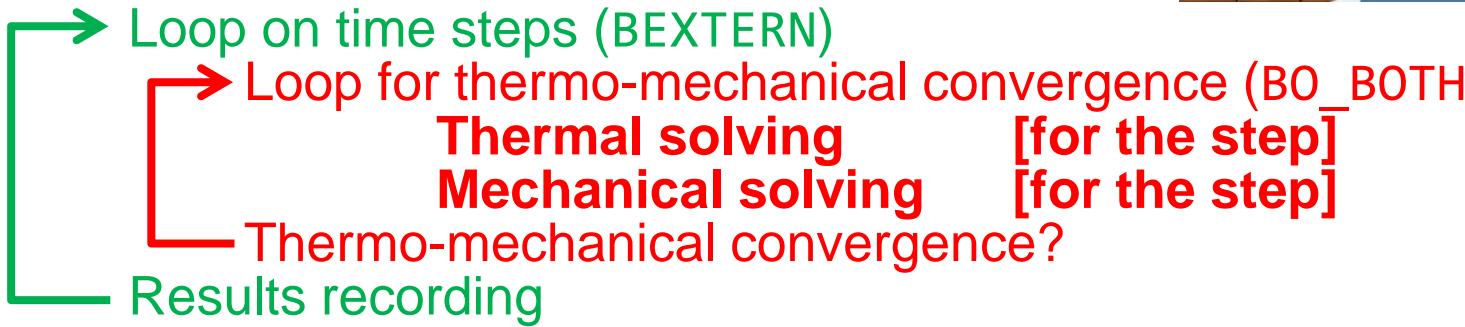
EV1 = EVOL 'TEMP' TAB1 'DEPLACEMENTS' 'UX' P1 ;

PASAPAS OPERATION

PASAPAS OPERATION

■ Main algorithm

Initializations



End

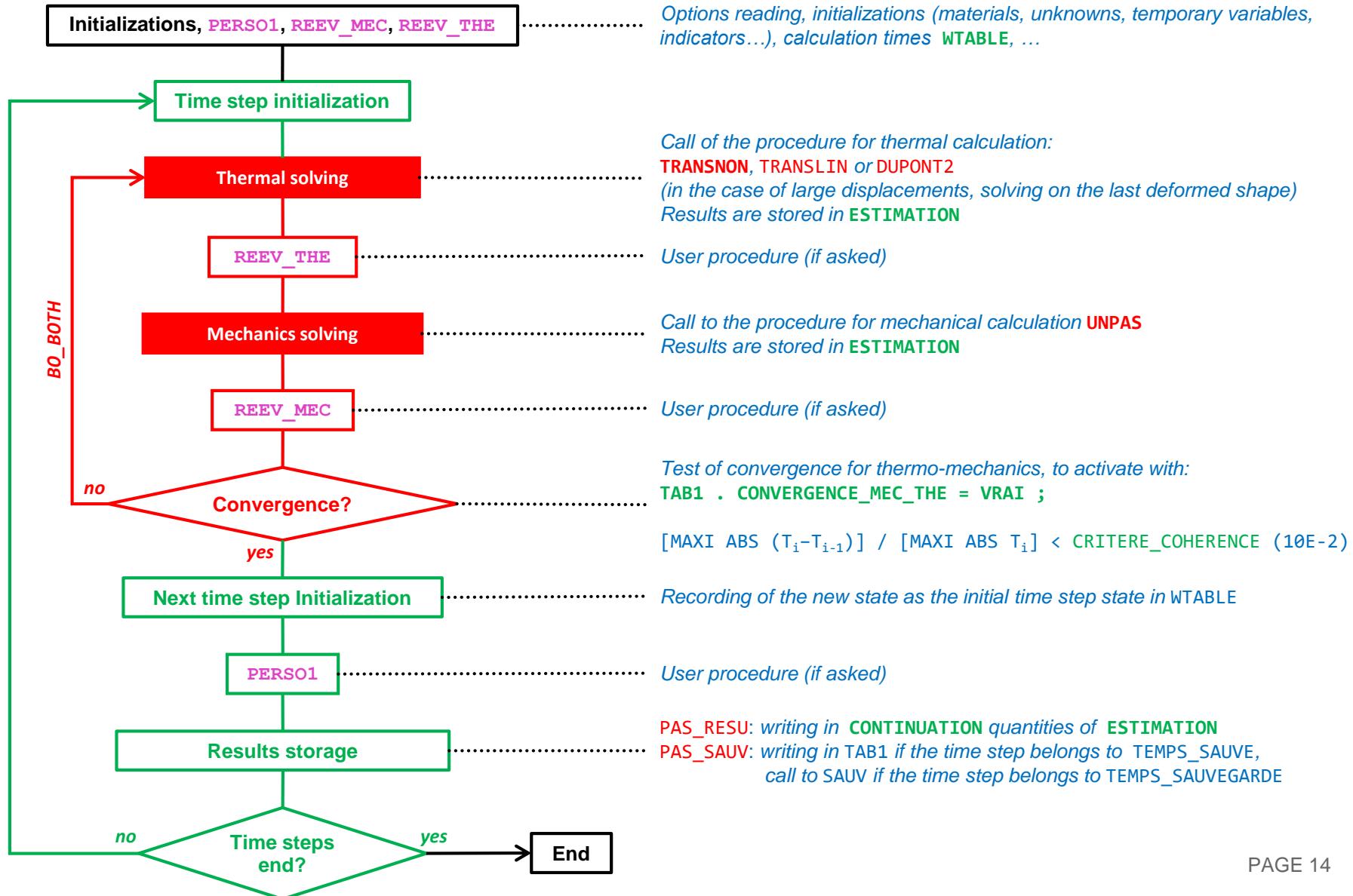
+ User procedures calls (optional, user have to ask it)

**PERSO1 PERSO2 REEV_MEC REEV_THE
CHARMECA CHAROTHER PARATHER**

procedures to be defined by the user!



PASAPAS OPERATION



ACCESS TO PASAPAS DATA

■ TAB1 . 'ESTIMATION'

contains all results calculated/converged by TRANSNON and UNPAS
but not converged for BO_BOTH loop

```
TAB1 . 'ESTIMATION' . 'TEMPS'  
TAB1 . 'ESTIMATION' . 'DEPLACEMENTS'  
TAB1 . 'ESTIMATION' . 'CONTRAINTES'  
TAB1 . 'ESTIMATION' . 'TEMPERATURES'
```

...

...

■ TAB1 . 'CONTINUATION'

contains converged results (for BO_BOTH loop)
this index is updated at the **end of the time-step!!!**

➔ useful for PASAPAS restart

```
TAB1 . 'CONTINUATION' . 'TEMPS'  
TAB1 . 'CONTINUATION' . 'DEPLACEMENTS'  
TAB1 . 'CONTINUATION' . 'CONTRAINTES'  
TAB1 . 'CONTINUATION' . 'TEMPERATURES'
```

...

...

ACCESS TO PASAPAS DATA

■ TAB1 . 'WTABLE'

Contains all useful variables for PASAPAS (*chosen options, models, materials and instant loading, intermediate results ...*)

Some indexes:

WTABLE . 'CHARGEMENT'

Loading on the time step

WTABLE . 'THER_COURANT'

Value of temperature at last iteration (during a time step)

WTABLE . 'BLOCAGES_MECANIQUES'

Mechanical constraint matrix

WTABLE . 'BLOCAGES_THERMIQUES'

Thermal constraint matrix

WTABLE . 'FOR'

Configuration at the time step beginning

WTABLE . 'FOR0'

Initial Configuration

WTABLE . 'MODELE'

Models

WTABLE . 'CARACTERISTIQUES'

Material properties

...

For more information, see:

→ Comments in procedure PAS_DEF A (http://www-cast3m.cea.fr/index.php?page=procedures&procedure=pas_defa)

→ The PASAPAS documentation (http://www-cast3m.cea.fr/html/Documentation_Cast3M/Pasapas.pdf)

FINDING THE MECHANICAL EQUILIBRIUM

THE UNPAS PROCEDURE

REMINDER ON EQUATIONS

■ Local equations of the static equilibrium

equilibrium	$\operatorname{div}(\boldsymbol{\sigma}) + f = 0$	on V
prescribed surface forces	$\boldsymbol{\sigma} \cdot \mathbf{n} = t$	on ∂V^t
prescribed displacements	$u = d$	on ∂V^d



■ Weak form + FE discretization

$$\{F\} - \int_V [\mathbf{B}]^T \{\boldsymbol{\sigma}\} dV = \{0\}$$

$$\underbrace{\int_{\partial V^t} [N]^T \{t\} dS}_{\{F\}^S} + \underbrace{\int_{\partial V^d} [N]^T \{\boldsymbol{\sigma} \cdot \mathbf{n}\} dS}_{\{F\}^R} + \underbrace{\int_V [N]^T \{f\} dV}_{\{F\}^V} - \underbrace{\int_V [\mathbf{B}]^T \{\boldsymbol{\sigma}\} dV}_{[\mathbf{B}]\{\boldsymbol{\sigma}\}} = \{0\}$$

Vectors of Equivalent nodal forces (\mathbf{N})

$\{F\}^S$	to the prescribed surface forces t (Pa)	(PRES, FSUR, FORC, ...)
$\{F\}^R$	to the surface forces in reaction to the prescribed displacements d (m)	(REAC)
$\{F\}^V$	to the prescribed volume forces f (N.m ⁻³)	(CNEQ)
$[\mathbf{B}]\{\boldsymbol{\sigma}\}$	to the internal volume forces	(BSIG)

Matrices

$[N]$	matrix of shape (interpolation) functions
$[B]$	matrix of derivatives of shape functions

IMBALANCE

- Imbalance = measure of deviation of equilibrium

$$\{R\} = \underbrace{\{F\}^S + \{F\}^R + \{F\}^V}_{\{F\}^{ext}} - \underbrace{[B]\{\sigma\}}_{\{F\}^{int}}$$

- The balance is obtained when

$$\|R\| < \varepsilon \cdot F^{ref}$$

$\|R\|$ norm of the imbalance vector, for instance infinity norm
 ε computational precision (given by user)
 F^{ref} reference force for the studied problem

NON LINEAR BEHAVIOR

■ Introduction of the linear elastic stiffness matrix

$$\boldsymbol{\sigma} = \mathbb{E}: \boldsymbol{\varepsilon}^{lin} + \boldsymbol{\sigma}^{nl} \quad \text{with } \boldsymbol{\varepsilon}^{lin} = \frac{1}{2}(\nabla u + \nabla^T u) \\ \{\varepsilon\}^{lin} = [B]\{U\}$$

■ Equilibrium can be re written

$$[B]\{\sigma\} = \{F\}^S + \{F\}^R + \{F\}^V \\ [B][E]\{\varepsilon\}^{lin} = \{F\}^S + \{F\}^R + \{F\}^V - [B]\{\sigma\}^{nl}$$

$$\underbrace{\int_V [B]^T [E] [B] dV}_{[K]} \{U\} = \{F\}^S + \{F\}^R + \{F\}^V - [B]\{\sigma\}^{nl}$$

$$[K]\{U\} = \{F\}^S + \{F\}^R + \{F\}^V - [B]\{\sigma\}^{nl}$$

$\{U\}$ nodal displacements

$u(x) = [N(x)]\{U\}$

$\{\varepsilon\}^{lin}$ linear strains (« small » strains)

$\{\varepsilon\}^{lin} = [B]\{U\}$ (**EPSI**)

$\{\sigma\}^{nl}$ complementary non linear stresses

$[E]$ Hooke matrix (**ELAS**)

$[K]$ elastic stiffness matrix (**RIGI**)

INTRODUCING THE PRESCRIBED DISPLACEMENTS

■ Boundary condition matrix

$$u = d \quad \text{on } \partial V^d$$

$$[A]\{U\} = \{d\}$$

■ Lagrange Multipliers

For a lin. system without prescribed displacements: $[K]\{U\} = \{F\}^S + \{F\}^V - [B]\{\sigma\}^{nl}$

We add new unknowns $\{\lambda\}$

$$[K]\{U\} + \underbrace{[A]^T\{\lambda\}}_{-\{F\}^R} = \{F\}^S + \{F\}^V - [B]\{\sigma\}^{nl} \quad \text{with } [A].\{U\} = \{d\}$$

Equilibrium is now written:

$$\underbrace{\begin{bmatrix} K & A^T \\ A & 0 \end{bmatrix}}_{[\hat{K}]}\begin{bmatrix} U \\ \lambda \end{bmatrix} = \begin{bmatrix} F^S + F^V - [B]\{\sigma\}^{nl} \\ d \end{bmatrix}$$

depends on $\{U\}$!!
 non linear behavior,
 large displacements,
 following forces,
 ...

- $[A]$ boundary condition matrix (**BLOQ, RELA**)
- $\{d\}$ prescribed nodal displacements (**DEPI**)
- $\{\lambda\}$ Lagrange multipliers

INCREMENTAL DECOMPOSITION (1/2)

■ Computation on a time step between t_0 and t_1

$[t_0, \{U\}_0, \{\lambda\}_0, \{\sigma\}_0]$ known state beginning of time step

$[t_1, \{U\}_1, \{\lambda\}_1, \{\sigma\}_1]$ unknown state end of time step

■ Incremental decomposition of nodal displacements

$$\{U\}_1^i = \{U\}_0 + \Delta\{U\}_1^i \quad \text{and} \quad \Delta\{U\}_1^{i+1} = \Delta\{U\}_1^i + \delta\{U\}_1^{i+1}$$

and so

$$\{U\}_1^{i+1} = \{U\}_1^i + \delta\{U\}_1^{i+1}$$

$\{U\}_1^i$ estimation of displacement $\{U\}_1$ at iteration i

$\Delta\{U\}_1^i$ estimation of displacement increment at iteration i

$\delta\{U\}_1^{i+1}$ correction of displacement increment at iteration i

■ Incremental decomposition of boundary conditions

$$[A]\{U\}_1^{i+1} = \{d\}_1$$

$$[A]\delta\{U\}_1^{i+1} = \{d\}_1 - [A]\{U\}_1^i$$

INCREMENTAL DECOMPOSITION (2/2)

■ Incremental decomposition of equilibrium

$$[K]\{U\}_1^{i+1} + [A]^T\{\lambda\}_1^{i+1} = \{F\}_1^S + \{F\}_1^V - [B]\{\sigma\}_1^{nl,i}$$

$$[K]\delta\{U\}_1^{i+1} = \{F\}_1^S + \{F\}_1^V - [A]^T\{\lambda\}_1^{i+1} - \underbrace{([K]\{U\}_1^i + [B]\{\sigma\}_1^{nl,i})}_{[B]\{\sigma\}_1^i}$$

$$[K]\delta\{U\}_1^{i+1} = \underbrace{\{F\}_1^S + \{F\}_1^V - [A]^T\{\lambda\}_1^{i+1}}_{\{R\}_1^i} - [B]\{\sigma\}_1^i$$

■ Equilibrium is finally written

$$\boxed{\underbrace{\begin{bmatrix} K & A^T \\ A & 0 \end{bmatrix}}_{[\hat{K}]}\begin{Bmatrix} \delta U_1^{i+1} \\ \lambda_1^{i+1} \end{Bmatrix} = \begin{Bmatrix} F_1^S + F_1^V - [B]\{\sigma\}_1^i \\ \{d\}_1 - [A]\{U\}_1^i \end{Bmatrix}}$$

IMBALANCE MINIMIZATION ALGORITHM

■ Simple algorithm

$$[\{U\}_1^{i=0}, \{\lambda\}_1^{i=0}, \{\sigma\}_1^{i=0}] = [\{U\}_0, \{\lambda\}_0, \{\sigma\}_0]$$

$$\{F\}_1^{R,i=0} = -[A]^T \{\lambda\}_1^{i=0}$$

$$F^{ref} = \|\{F\}_1^S + \{F\}_1^V + \{F\}_1^{R,i=0}\|$$

$$\{R\}_1^{i=0} = \{F\}_1^S + \{F\}_1^V + \{F\}_1^{R,i=0} - [B]\{\sigma\}_1^{i=0}$$

initializations

reaction forces (**REAC**)

convergence norm (**MAXI 'ABS'**)

first imbalance (residual) (**BSIG**)

While: $\|\{R\}_1^i\|/F^{ref} \geq \varepsilon$

$$[\delta\{U\}_1^{i+1}, \{\lambda\}_1^{i+1}] = [\hat{K}]^{-1} \{R\}_1^i$$

$$\{U\}_1^{i+1} = \{U\}_1^i + \delta\{U\}_1^{i+1}$$

$$\{\varepsilon\}_1^{i+1} = \mathcal{D}(\{U\}_1^{i+1})$$

$$\{\sigma\}_1^{i+1} = \mathcal{C}(\Delta\{\varepsilon\}_1^{i+1})$$

$$\{F\}_1^{R,i+1} = -[A]^T \{\lambda\}_1^{i+1}$$

$$\{R\}_1^{i+1} = \{F\}_1^S + \{F\}_1^V + \{F\}_1^{R,i+1} - [B]\{\sigma\}_1^{i+1}$$

$$i = i + 1$$

resolution (**RESO**)

estim. displacements

estim. strains (**EPSI**)

estim. stresses (**COMP**)

estim. reaction forces (**REAC**)

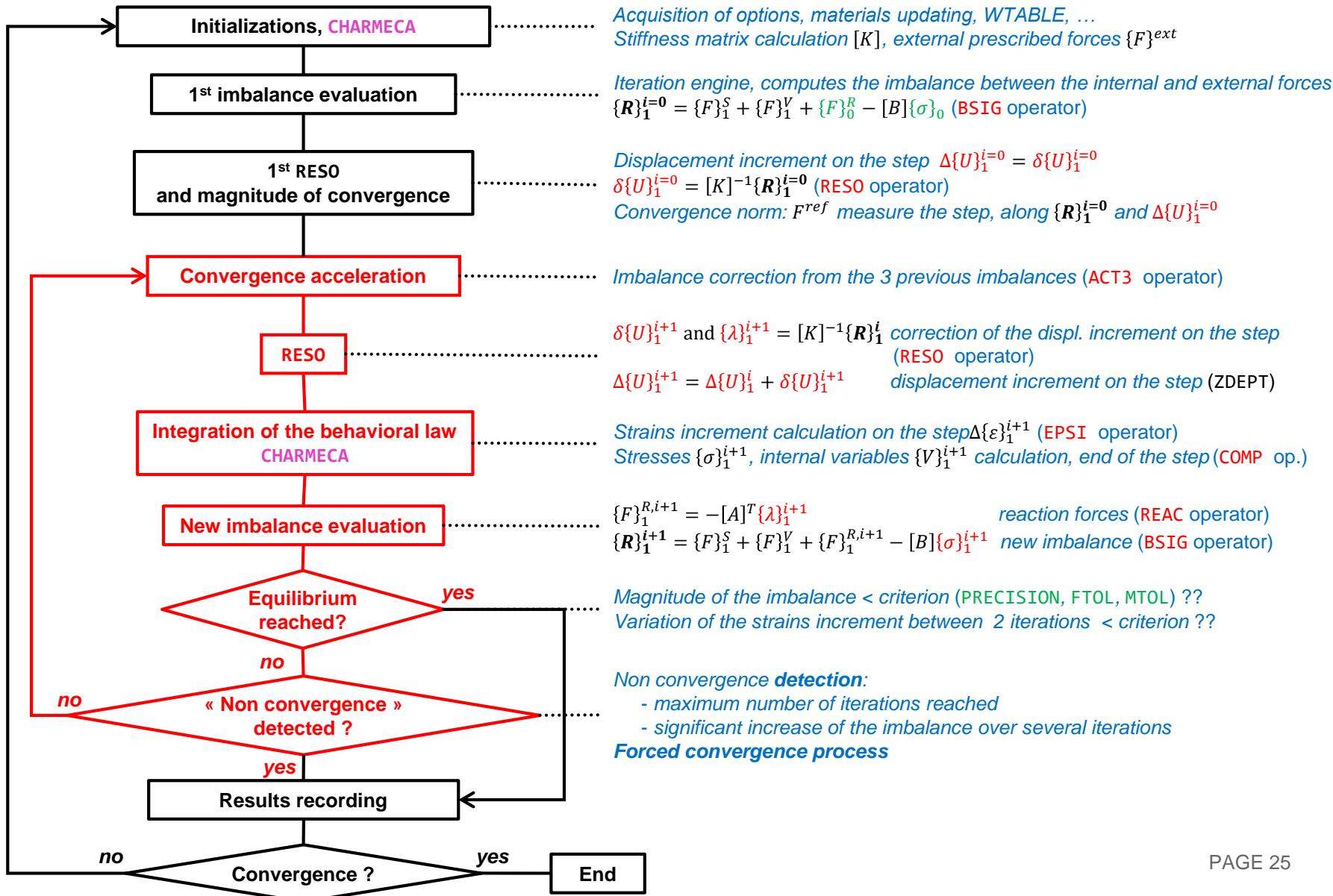
new imbalance (residual) (**BSIG**)



End

UNPAS OPERATION

Forced Convergence:
Restart from the out of unbalanced state



USERS PROCEDURES IN PASAPAS

PERSO1 REEV_MEC CHARMEECA

PERSO2 REEV_THE CHARTHER PARATHER

USERS PROCEDURES: INSTRUCTIONS FOR USE

- There exists 7 users procedures called at different stages of PASAPAS algorithm

PERS01
CHARMECA

PERS02
CHARTHER

REEV_MEC
PARATHER

REEV_THE

- 1) According to the desired action, choose the procedure to be used as a function of its location on the algorithm

for instance: since PERS01 is called after computing a time step it is dedicated to update the next time step data
e.g. boundary conditions, material parameters, ...

- 2) Analyze the procedure syntax

- in the PASAPAS manual pages
- In the PASAPAS, UNPAS or TRANSNON procedures

the following table recalls the procedures syntaxes

PERSO1 PROCEDURE EXAMPLE

■ 3) Define the desired procedure

```
DEBP PERSO1 T1*'TABLE' ;
MESS 'Hello! The PERSO1 procedure is talking to you.' ;
FINP ;
```

■ 4) Indicate in the PASAPAS table that this procedure should be called

```
TAB1 . 'MODELE' = MOD1 ;
TAB1 . 'CARACTERISTIQUES' = MAT1 ;
TAB1 . 'TEMPS_CALCULES' = PROG 1. 2. 3. 4. ;
TAB1 . 'PROCEDURE_PERSO1' = VRAI ;
...
...
PASAPAS TAB1 ;
```

USERS PROCEDURES LIST

Name	Index to be activated in PASAPAS table	Syntax	Function
PERS01	TAB1 . 'PROCEDURE_PERS01' = VRAI ;	PERS01 T1 ;	Updates the problem after the mechanical calculation step
REEV_MEC	TAB1 . 'PROCEDURE_REEV_MEC' = VRAI ;	REEV_MEC T1 N1 ;	Ditto (but in BO_BOTH) → the step can be started again
CHARMeca	TAB1 . 'PROCEDURE_CHARMeca' = VRAI ;	T2 = CHARMeca T1 TPS1 ;	Mechanical loads addition during UNPAS iterations
PERS02	TAB1 . 'PROCEDURE_PERS02' = VRAI ;	PERS02 T1 ;	Updates the problem after the thermal calculation step
REEV_THE	TAB1 . 'PROCEDURE_REEV_THE' = VRAI ;	REEV_THE T1 N1 ;	Ditto (but in PASAPAS) → the step can be started again
CHARther	TAB1 . 'PROCEDURE_CHARther' = VRAI ;	T2 = CHARther T1 TPS1 ;	Thermal loads addition during TRANSNON iterations
PARATHER	TAB1 . 'PROCEDURE_PARATHER' = VRAI ;	PARATHER T1 TPS1 ;	Updating of the external variables of thermal parameters

With

T1 : the PASAPAS table

TPS1 : the current calculated time

N1 : the call number of the procedure (0 or 1) → see the PASAPAS procedure

T2 : the output table (only for the CHARMECA and CHARther procedures)

A FEW REMARKS

- Instructions inside these procedures are **free !!!**
- **CHARMECA** and **CHARTHER** output is a table with 2 possible indexes:
 - '**ADDI_MATRICE**' contains the stiffness matrices (RIGIDITE type object) to be added to the 1st member
 - '**ADDI_SECOND**' contains the CHPOINT to be added to the 2nd member (nodal forces)
- For large displacements ('**GRANDS_DEPLACEMENTS**' option), **CHARMECA** is called on the deformed shape

A SIMPLE EXAMPLE

■ Random variation of Young's modulus map at each time step

```

OPTI 'DIME' 2 'ELEM' 'QUA4' ;
P0 = 0. 0. ; P1 = 1. 0. ;
L1 = DROI 1 P0 (0. 0.1) ;
S1 = L1 TRAN 10 (1. 0.) ;
L2 = S1 COTE 3 ; P2 = L2 POIN 'PROC' (1. 0.) ;

MO = MODE S1 'MECANIQUE' ;
MA = MATE MO 'YOUN' 200.E9 'NU' 0.3 ;

BL = (BLOQ 'UX' L1) ET (BLOQ 'UY' P0) ;
F = PRES 'MASS' MO L2 -1.E8 ;
EV = EVOL 'MANU' (PROG 0. 1.) (PROG 0. 1.) ;
CHA = CHAR 'MECA' F EV ;

TAB1 = TABL ;
TAB1 . 'MODELE' = MO ;
TAB1 . 'CARACTERISTIQUES' = MA ;
TAB1 . 'BLOCAGES_MECANIQUES' = BL ;
TAB1 . 'CHARGEMENT' = CHA ;
TAB1 . 'TEMPS_CALCULES' = PROG 0.1 'PAS' 0.1 1. ;
TAB1 . 'PROCEDURE_PERSO1' = VRAI ;

PASAPAS TAB1 ;

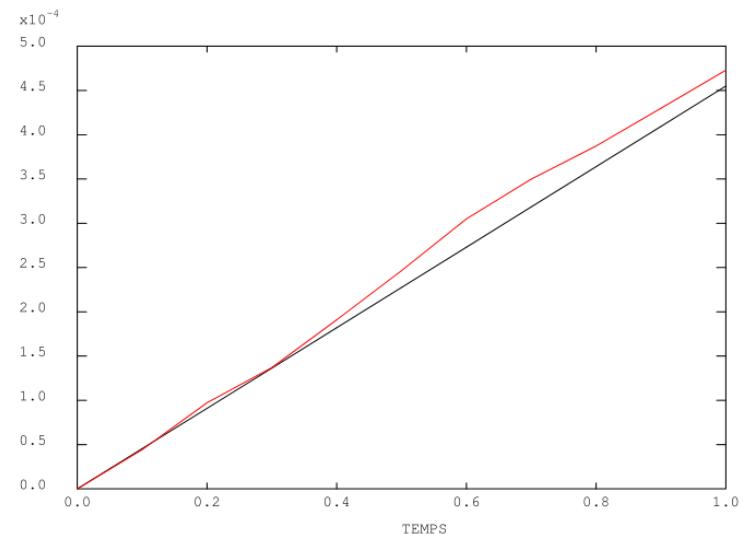
EVU = EVOL 'TEMP' TAB1 'DEPLACEMENTS' 'UX' P2 ;
DESS EVU ;

```

```

DEBP PERSO1 T1*'TABLE' ;
MOD = T1 . 'WTABLE' . 'MO_TOT' ;
MAIL = EXTR MOD 'MAIL' ;
CHPYO = BRUI 'BLAN' 'UNIF' 200.E9 150.E9 MAIL ;
CHMYO = CHAN 'CHAM' CHPYO MOD 'RIGIDITE' ;
MAT = MATE MOD 'YOUN' CHMYO 'NU' 0.3 ;
T1 . 'WTABLE' . 'CARACTERISTIQUES' = MAT ;
FINP ;

```



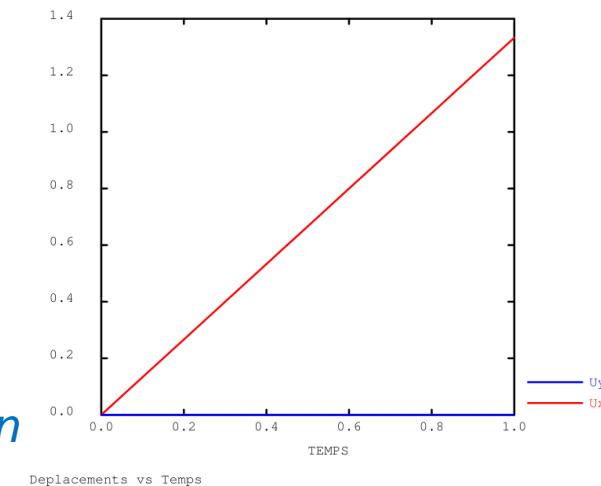
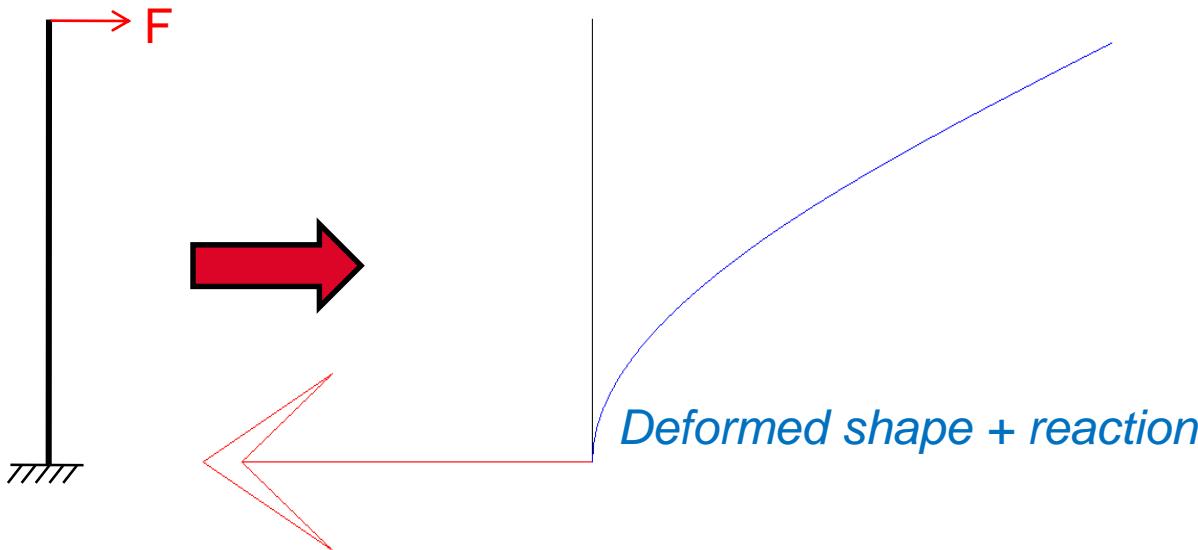
EXERCISE 1: BEAM WITH FOLLOWING FORCE

DOWNLOAD THE STARTING FILE ON THE WEBSITE:

[HTTP://WWW-CAST3M.CEA.FR/INDEX.PHP?PAGE=EXEMPLES&EXEMPLE=FORMATION_PASAPAS_1_INITIAL](http://WWW-CAST3M.CEA.FR/INDEX.PHP?PAGE=EXEMPLES&EXEMPLE=FORMATION_PASAPAS_1_INITIAL)

EXERCISE 1: BEAM WITH FOLLOWING FORCE

- Beam bending
base is clamped
point shear force at the top of the beam (perpendicular to the beam)
large displacement at the top of the beam



- Problem: the force is calculated on the initial shape and not updated
- Purpose: to correctly apply the force during the calculation

It's up to you!

EXERCISE 1: BEAM WITH FOLLOWING FORCE

A few indications ...

■ Useful objects

- P2 : point at the top of the beam, where the force is prescribed
EV1 : force magnitude to be applied as a function of time

■ Useful operators

EXTR, COS, SIN

IPOL : to interpolate the force magnitude as a function of the calculated step

FORC : to apply a point force

EXERCISE 1: BEAM WITH FOLLOWING FORCE

Solution

- use option '**GRANDS_DEPLACEMENTS**'
(equilibrium verified on the deformed shape)
- use procedure **PERSO1** (1 call / time step)
to re-calculate the force on deformed shape (at beginning of time step)
- create a "load" with it (CHARGEMEnT type object)
- overwrite the load in **WTABLE**

```

DEBP PERSO1 T1*'TABLE' ;
* donnees utiles
TAM = T1 . 'MY_DATA' ;
PF = TAM . 'POINT' ;
EV = TAM . 'EVOL' ;
* angle de rotation de la section droite
U1 = T1 . 'ESTIMATION' . 'DEPLACEMENTS' ;
RS = EXTR U1 'RZ' PF ;
RSD = RS * 180. / PI ;
* direction de l'effort
DIR1 = (COS RSD) (SIN RSD) ;
* creation du second membre
F1 = FORC DIR1 PF ;
CHAR2 = CHAR 'MECA' F1 EV ;
T1 . 'WTABLE' . 'CHARGEMENT' = CHAR2 ;
FINP ;

```

PERSO1 procedure

```

TAB1 . 'GRANDS_DEPLACEMENTS' = VRAI ;
TAB1 . 'PREDICTEUR'          = 'HPP' ;
TAB1 . 'PROCEDURE_PERSO1'    = VRAI ;
TAB1 . 'MY_DATA'             = TABL ;
TAB1 . 'MY_DATA' . 'EVOL'    = EV1 ;
TAB1 . 'MY_DATA' . 'POINT'   = P2 ;

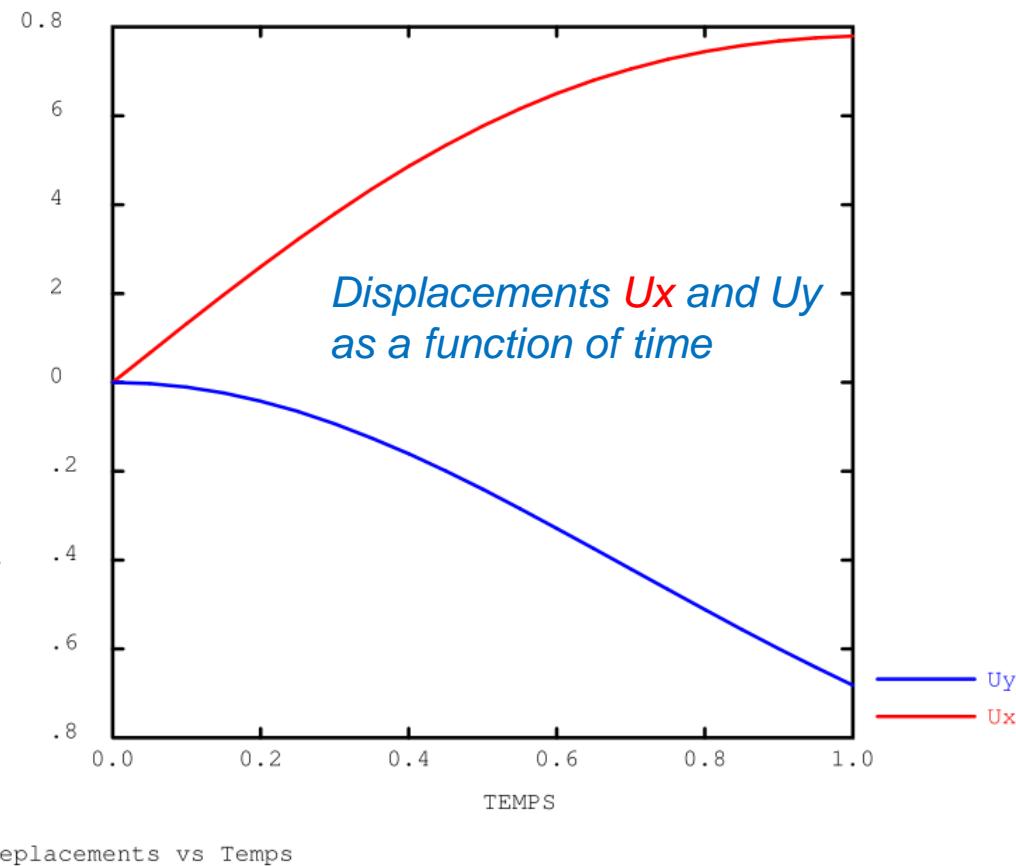
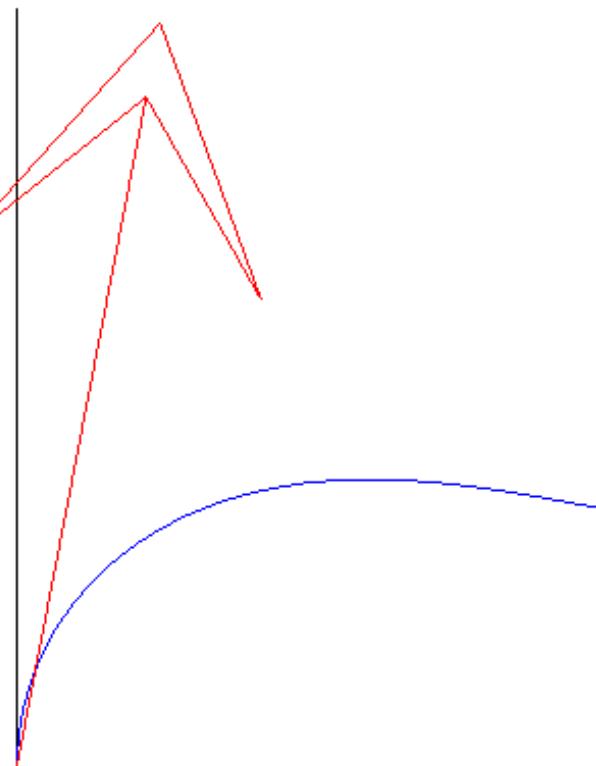
```

Main program

EXERCISE 1: BEAM WITH FOLLOWING FORCE

■ Results

Deformed shape + reaction



EXERCISE 1: BEAM WITH FOLLOWING FORCE

Solution (bis)

- idem but with the **CHARMECA** procedure
- suppress initial mechanical loading
- no need of CHARGEMEnt type object
- longer: 1 call / each iteration / time step
- results are identical

```

DEBP CHARMECA T1*'TABLE' TPS1*'FLOTTANT' ;
* donnees utiles
TAM = T1 . 'MY_DATA' ;
PF = TAM . 'POINT' ;
EV = TAM . 'EVOL' ;
* angle de rotation de la section droite
U1 = T1 . 'ESTIMATION' . 'DEPLACEMENTS' ;
RS = EXTR U1 'RZ' PF ;
RSD = RS * 180. / PI ;
* direction de l'effort
DIR1 = (COS RSD) (SIN RSD) ;
* creation du second membre
VAL1 = IPOL TPS1 EV ;
F1 = FORC (VAL1 * DIR1) PF ;
T2 = TABLE ;
T2 . 'ADDI_SECOND' = F1 ;
FINP T2 ;

```

CHARMECA procedure

```

***TAB1 . 'CHARGEMENT'      = CHAR1 ;
TAB1 . 'GRANDS_DEPLACEMENTS' = VRAI ;
TAB1 . 'PREDICTEUR'          = 'HPP' ;
TAB1 . 'PROCEDURE_CHARMECA' = VRAI ;
TAB1 . 'MY_DATA'             = TABL ;
TAB1 . 'MY_DATA' . 'EVOL'    = EV1 ;
TAB1 . 'MY_DATA' . 'POINT'   = P2 ;

```

Main program

EXERCISE 1: BEAM WITH FOLLOWING FORCE

Solution (ter)

- idem, but the force is updated on the deformed shape at the end of time step!
- **more precise:** when converged, the deformed geometry is exactly perpendicular to applied force
- **watch out: can become unstable** 🤦 !

```

DEBP CHARMECA T1*'TABLE' TPS1*'FLOTTANT' ;
* donnees utiles
TAM = T1 . 'MY_DATA' ;
PF = TAM . 'POINT' ;
EV = TAM . 'EVOL' ;
* angle de rotation de la section droite
DRS = 0. ;
SI (EGA (TYPE ZDEPT) 'CHPOINT') ;
  DRS = EXTR ZDEPT 'RZ' PF ;
FINSI ;
U0 = T1 . 'CONTINUATION' . 'DEPLACEMENTS' ;
RS = (EXTR U0 'RZ' PF) + DRS ;
RSD = RS * 180. / PI ;
* direction de l'effort
DIR1 = (COS RSD) (SIN RSD) ;
* creation du second membre
VAL1 = IPOL TPS1 EV ;
F1 = FORC (VAL1 * DIR1) PF ;
T2 = TABLE ;
T2 . 'ADDI_SECOND' = F1 ;
FINP T2 ;

```

CHARMECA procedure

```

***TAB1 . 'CHARGEMENT'           = CHAR1 ;
TAB1 . 'GRANDS_DEPLACEMENTS' = VRAI ;
TAB1 . 'PREDICTEUR'             = 'HPP' ;
TAB1 . 'PROCEDURE_CHARMECA'   = VRAI ;
TAB1 . 'MY_DATA'                = TABL ;
TAB1 . 'MY_DATA' . 'EVOL'      = EV1 ;
TAB1 . 'MY_DATA' . 'POINT'     = P2 ;

```

Main program

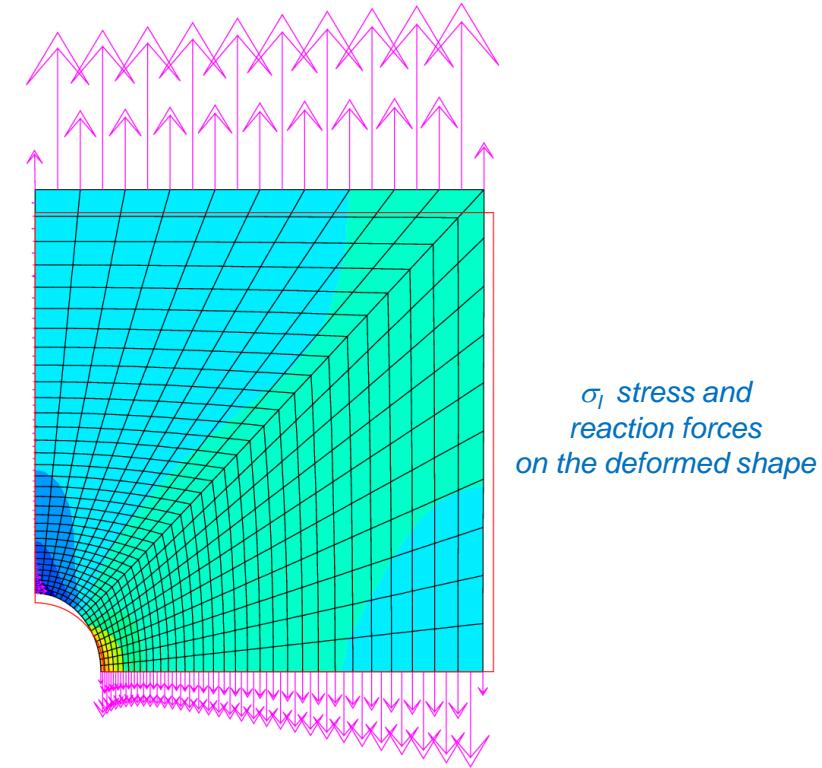
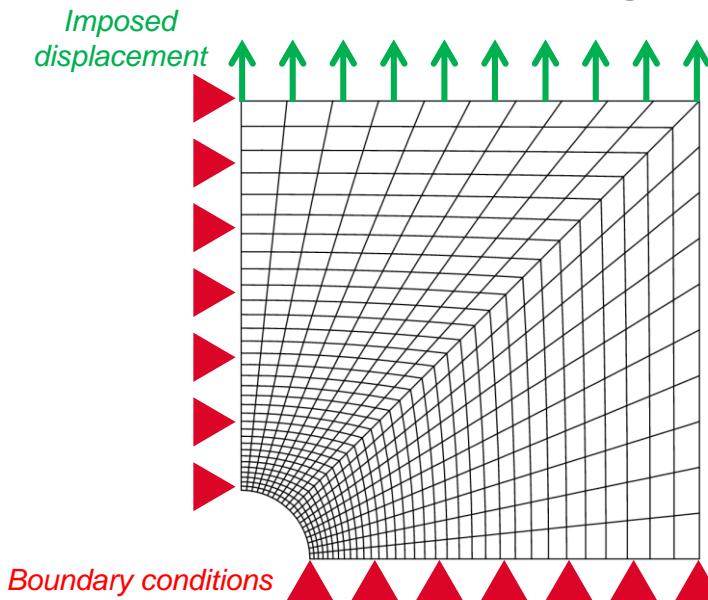
EXERCISE 2: RUPTURE BY ELEMENTS REMOVAL

DOWNLOAD THE STARTING FILE ON THE WEBSITE:

[HTTP://WWW-CAST3M.CEA.FR/INDEX.PHP?PAGE=EXEMPLES&EXAMPLE=FORMATION_PASAPAS_2_INITIAL](http://WWW-CAST3M.CEA.FR/INDEX.PHP?PAGE=EXEMPLES&EXAMPLE=FORMATION_PASAPAS_2_INITIAL)

EXERCISE 2: RUPTURE BY ELEMENTS REMOVAL

- **Traction of a perforated plate**
elastic behavior, large displacements



- **Purpose: to model fracture by removing elements during calculations**

*We will use a simple criterion based on the 1st principal stress:
fracture if $\sigma_1 \geq 22 \text{ GPa}$*

It's up to you!

EXERCISE 2: RUPTURE BY ELEMENTS REMOVAL

A few indications ...

■ Useful objects

PRIN: principal stress field calculations

CHAN: change the support points of a field by elements

ELEM: select the elements of a field that meets a criteria

REDU: reduction of a model on a sub-mesh

■ Useful information

Modify the **temporary objects** for computation in **TAB1 . WTABLE**

EXERCISE 2: RUPTURE BY ELEMENTS REMOVAL

Solution

- reduce time steps
- use the **PERS01** procedure
- extract the model and the calculated stresses (**T1 . ESTIMATION**)
- calculate the principal stresses
- determine the mesh of “intact elements”
- reduce the model on this mesh
- overwrite the model in the working table **WTABLE**

```

DEBP PERS01 T1*'TABLE' ;
MO1 = T1 . 'WTABLE' . 'MODELE' ;
SG1 = T1 . 'ESTIMATION' . 'CONTRAINTES' ;
SGP1 = PRIN SG1 MO1 ;
SG11 = CHAN 'GRAVITE' (EXCO 'SI11' SGP1) MO1 ;
MAIL2 = SG11 ELEM 'INFERIEUR' 2.2E10 ;
MO2 = REDU MO1 MAIL2 ;
T1 . 'WTABLE' . 'MODELE' = MO2 ;
FINP ;

```

PERS01 procedure

```

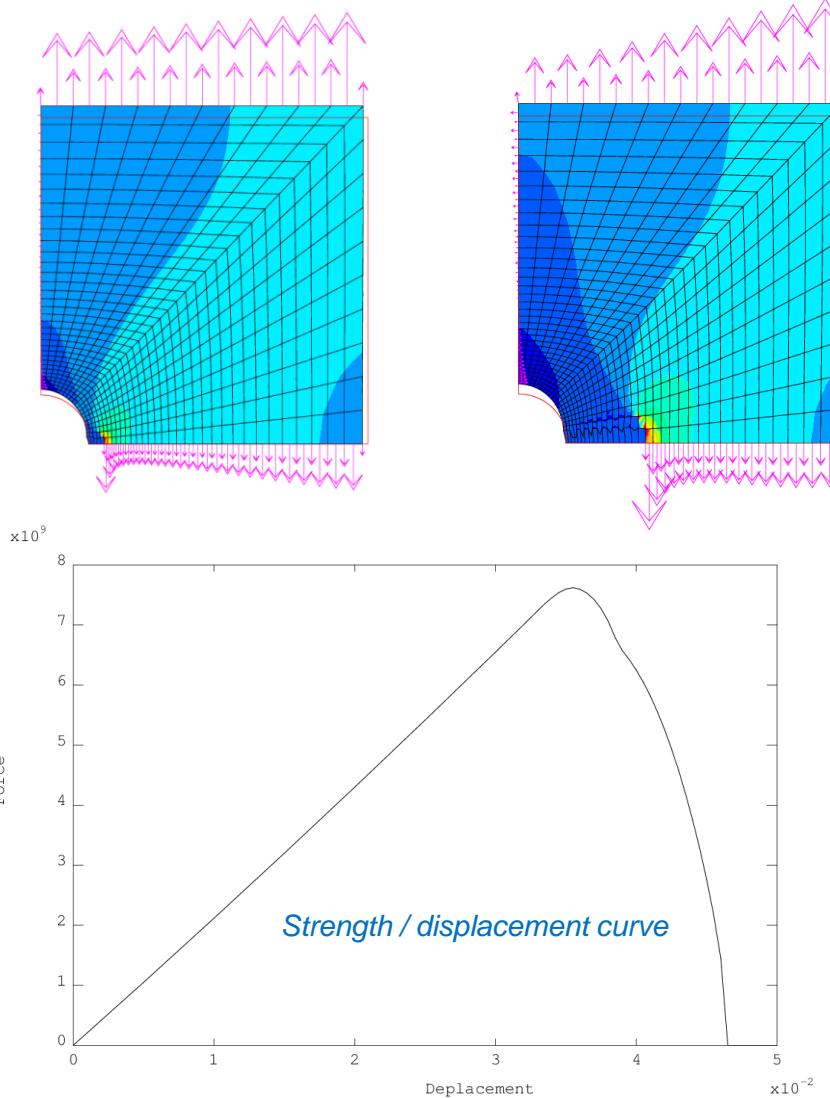
TAB1 . 'PROCEDURE_PERS01' = VRAI ;
TAB1 . 'TEMPS_CALCULES' = PROG 0.1 'PAS' 0.1 0.6
                                         'PAS' 0.01 1. ;

```

Main program

EXERCISE 2: RUPTURE BY ELEMENTS REMOVAL

■ Results



Undependable model:
results quite sensitive to time/space discretization

EXERCISE 2: RUPTURE BY ELEMENTS REMOVAL

Solution (bis)

- reduce time steps
- use the **PERS01** procedure
- extract the model and the calculated stresses (**T1 . ESTIMATION**)
- calculate the principal stresses
- determine the mesh of intact elements
- reduce the **boundary conditions** on this mesh
- overwrite the boundary conditions in the working table **WTABLE**

```

DEBP PERS01 T1*'TABLE' ;
MO1 = T1 . 'WTABLE' . 'MODELE' ;
MAIL1 = EXTR MO1 'MAIL' ;
SG1 = T1 . 'ESTIMATION' . 'CONTRAINTES' ;
SGP1 = PRIN SG1 MO1 ;
SG11 = CHAN 'GRAVITE' (EXCO 'SI11' SGP1) MO1 ;
MAIL2 = SG11 ELEM 'INFERIEUR' 2.2E10 ;
MAIL3 = DIFF MAIL1 MAIL2 ;
NE3 = NBEL MAIL3 ;
SI (NE3 > 0) ;
MESS '[PERS01 :] Removal of' NE3 'elements' ;
BL01 = T1 . 'WTABLE' . 'BLOCAGES_MECANIQUES' ;
MAILBL1 = EXTR BL01 'MAIL' ;
MAILBL2 = MAILBL1 ELEM 'APPU' 'LARG' MAIL3 ;
MAILBL3 = DIFF MAILBL1 MAILBL2 ;
BL02 = REDU BL01 MAILBL3 ;
T1 . 'WTABLE' . 'BLOCAGES_MECANIQUES' = BL02 ;
FINSI ;
FINP ;

```

PERS01 procedure

```

TAB1 . 'PROCEDURE_PERS01' = VRAI ;
TAB1 . 'TEMPS_CALCULES' = PROG 0.1 'PAS' 0.1 0.6
                                         'PAS' 0.01 1. ;

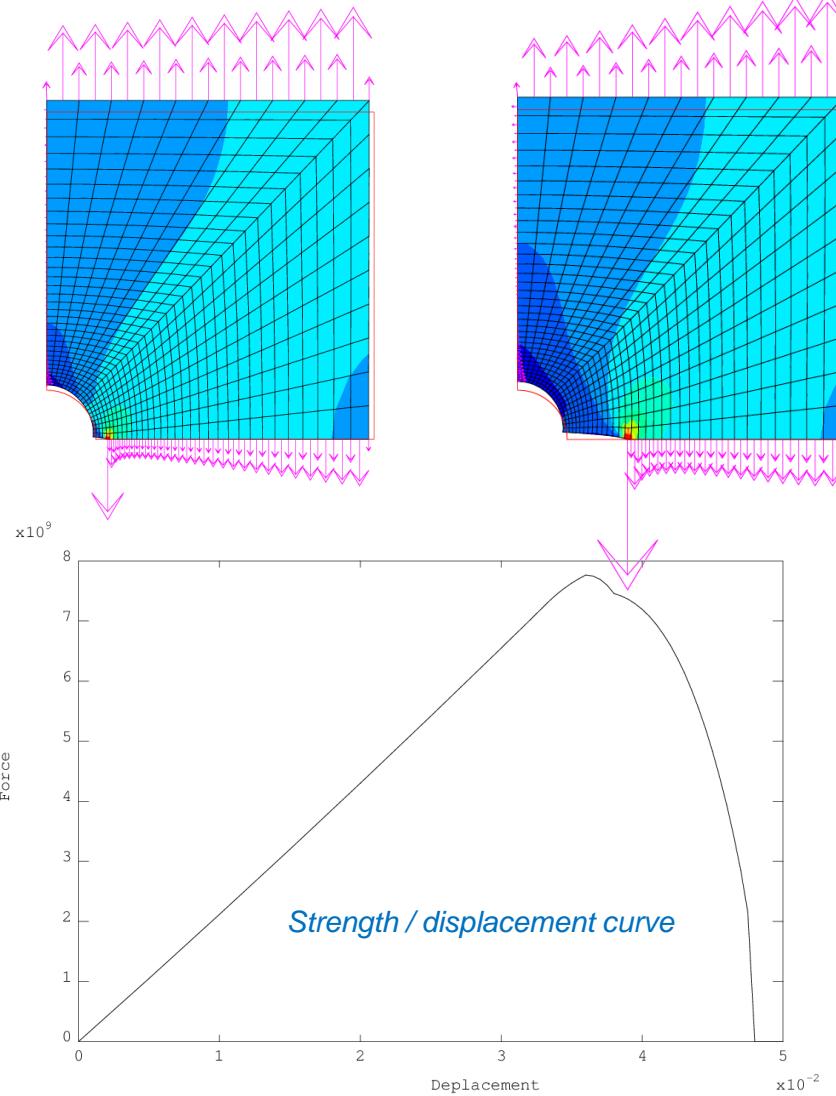
```

Main program

PAGE 44

EXERCISE 2: RUPTURE BY ELEMENTS REMOVAL

■ Results



Undependable model:
results quite sensitive to time/space discretization

TRANSNON OPERATION

THERMAL SOLVER

REMINDER ON EQUATIONS

■ Local equations in transient thermal analysis



heat flux density	$\vec{\varphi} = -\lambda \overrightarrow{\text{grad}}(T)$	on V
heat equation	$\rho c_p \frac{\partial T}{\partial t} + \text{div } \vec{\varphi} - q = 0$	on V
prescribed flux	$\vec{\varphi} \cdot \vec{n} = \varphi_{imp} + \underbrace{h(T_f - T)}_{\text{convection}} + \underbrace{\varepsilon\sigma(T_\infty^4 - T^4)}_{\text{radiation}}$	on ∂V^φ
prescribed temperature	$T = T_{imp}$	on ∂V^T

REMINDER ON EQUATIONS

■ Weak form + FE discretization

$$[C]\{\dot{T}\} + [K]\{T\} = \{F\}$$

Vectors of Equivalent Nodal Heat (W) $\{F\} = \{F\}^\varphi + \{F\}^V$

$\{F\}^\varphi = \int_{\partial V^\varphi} [N]^T \varphi_{imp} dS$ to the prescribed surface heat flux φ_{imp} (J.m⁻²) **(FLUX)**

$+ \int_{\partial V^\varphi} [N]^T h T_f dS$ to the prescribed convection flux $h T_f$ (J.m⁻²) **(CONV)**

$+ \int_{\partial V^\varphi} [N]^T \varepsilon \sigma (T_\infty^4 - T^4) dS$ to the prescribed radiation flux (J.m⁻²)

$\{F\}^V = \int_V [N]^T q dV$ to the prescribed volume heat q (J.m⁻³) **(SOUR)**

Matrices

$[C] = \int_V \rho c_p [N]^T [N] dV$ capacity matrix (J.K⁻¹) **(CAPA)**

$[K] = \int_V [B]^T [\lambda] [B] dV + \int_{\partial V^\varphi} h [N]^T [N] dS$ conductivity matrix (W.K⁻¹) **(COND)**

$[N]$ matrix of shape (interpolation) functions

$[B]$ matrix of derivatives of shape functions

THETA METHOD

■ Weak form + FE discretization

$$[C]\{\dot{T}\} + [K]\{T\} = \{F\}$$

■ Time discretization: theta method

$$[C]_* \frac{\{T\}_{n+1} - \{T\}_n}{\Delta t} + [K]_* \{T\}_* = \{F\}_*$$

with:

Δt time step

$\{T\}_n$ vector of nodal temperatures at time t_n

$\{T\}_{n+1}$ vector of nodal temperatures at time t_{n+1}

$[C]_*$, $[K]_*$ and $\{F\}_*$ are estimated:

$$\text{at time} \quad t_* = \theta t_{n+1} + (1 - \theta) t_n$$

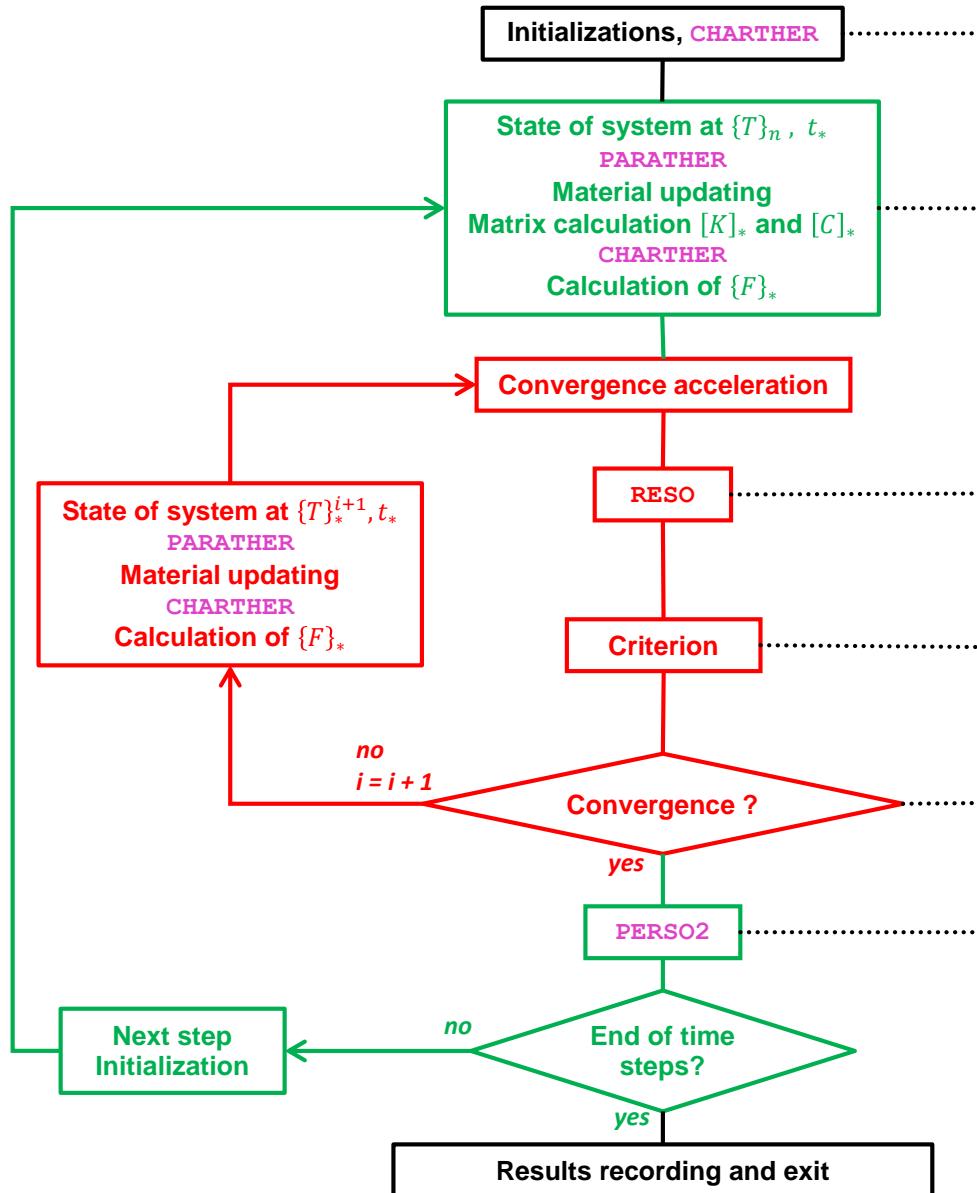
$$\text{at temperature} \quad \{T\}_* = \theta \{T\}_{n+1} + (1 - \theta) \{T\}_n$$

θ relaxation coefficient (between 0 and 1)

$\theta = 0$ explicit scheme

$\theta = 1$ implicit scheme (default value)

TRANSNON OPERATION



Options, times calculated, units ($^{\circ}\text{C}$ or K), initial state, shape functions (for radiation)...

External variables updating, material characteristics
 Calculation of conductivity matrix $[K]_*$ (cond. + conv. + radi. + constrain.) and capacity matrix $[C]_*$ at temperature $\{T\}_n$ and time t_*
 Calculation of 2nd member $\{F\}_*$ (T impo. + T conv. + T rayo. + Flux impo.) at temperature $\{T\}_n$ and time t_*

We define the operator as: $[L]_* = \frac{1}{\Delta t} [C]_* + \theta [K]_*$

Temperature increment calculation: $\Delta\{T\}^{i+1} = [L]_*^{-1} \{F\}_*$ and so we get: $\{T\}_{n+1}^{i+1} = \{T\}_n + \Delta\{T\}^{i+1}$ and also: $\{T\}_*^{i+1} = \theta \{T\}_{n+1}^{i+1} + (1 - \theta) \{T\}_n$

Measure the maximal difference on the temperature increment for two successive steps:
 $\text{Crit} = [\text{MAXI } 'ABS' (\Delta\{T\}^{i+1} - \Delta\{T\}^i)] / [\text{MAXI } 'ABS' \{T\}_{n+1}^{i+1}]$

If the criterion is small enough:
 $\text{Crit} \leq 'PRECISION'$ (1.E-4 by default)

User procedure

Remark:
 when TRANSNON is called by PASAPAS, the time step loop is performed once

EXERCISE 3: HEAT SOURCE DEPENDING ON TEMPERATURE

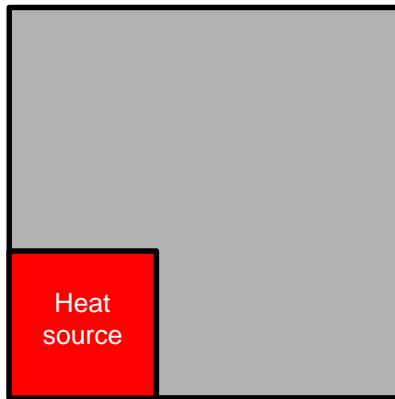
DOWNLOAD THE STARTING FILE ON THE WEBSITE:

[HTTP://WWW-CAST3M.CEA.FR/INDEX.PHP?PAGE=EXEMPLES&EXAMPLE=FORMATION_PASAPAS_3_INITIAL](http://WWW-CAST3M.CEA.FR/INDEX.PHP?PAGE=EXEMPLES&EXAMPLE=FORMATION_PASAPAS_3_INITIAL)

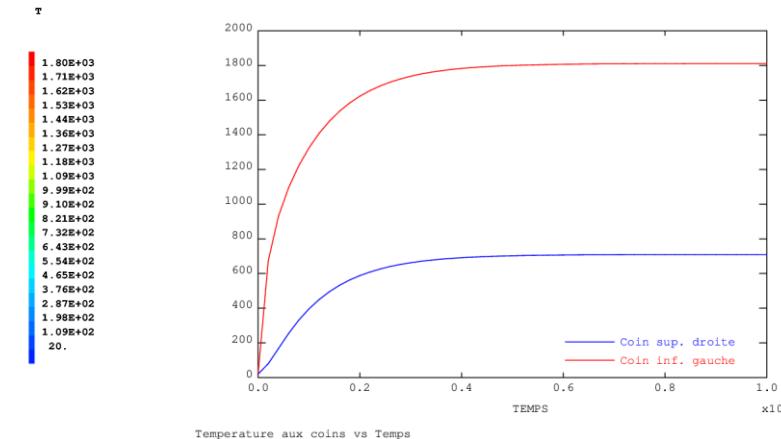
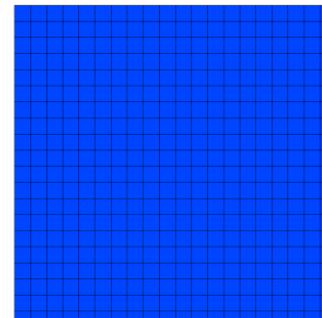
EXERCISE 3: HEAT GENERATION DEPENDING ON TEMPERATURE

■ Square section with heat source and cooled by convection

Convection



Convection



■ Purpose: make the problem depending on parameters

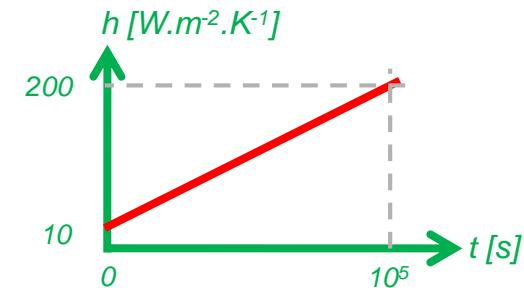
→ Conductivity as a function of temperature

$$\lambda(T) = 0.3 \cdot T + 200$$

→ Convection as a function of time

→ Source as a function of temperature

$$f(T) = 4\ 000\ 000 \ e^{-\left(\frac{T-1000}{700}\right)^2}$$



It's up to you!

EXERCISE 3: HEAT GENERATION DEPENDING ON TEMPERATURE

A few indications ...

■ Useful objects

MAIL1 : source mesh

MOD1 : thermal model reduced on MAIL1

■ Useful operators

REDU : to reduce the temperature field on the "source" area

SOUR : to impose a volumetric heat source

EXERCISE 3: HEAT GENERATION DEPENDING ON TEMPERATURE

Solution

- use the **PERSO2** procedure (called at each time step)
- recalculate the source (2nd member) as a function of temperatures at the beginning of the time step
- create a “load” with it and overwrite the load in **WTABLE**

```

DEBP PERSO2 T1*'TABLE' ;
MA1 = T1 . 'AMOI' . 'MAIL' ;
MO1 = T1 . 'AMOI' . 'MODE' ;
* reduction du champ de températures au maillage de la source
CHT1 = T1 . 'ESTIMATION' . 'TEMPERATURES' ;
CHT2 = REDU CHT1 MA1 ;
* calcul du champ de source à partir du champ de température
CHS = 4.E6 * (EXP (-1. * (((CHT2 - 1000.) / 700.) ** 2))) ;
* création du second membre et d'un chargement
CHFL1 = SOUR MO1 CHS ;
EV1 = EVOL 'MANU' (PROG 0. TPSMAX) (PROG 1. 1.) ;
CHA1 = CHAR 'Q' CHFL1 EV1 ;
* modification du chargement dans la table de calcul
T1 . 'WTABLE' . 'CHARGEMENT' = CHA1 ;
FINP T2 ;

```

PERSO2 procedure

```

LT      = PROG -1.E6 1.E6 ;
LL      = 200. + (0.3 * LT) ;
LAMBDA = EVOL 'MANU' 'T' LT 'K' LL ;

HCONV   = EVOL 'MANU' 'TEMP' (PROG 0. TPSMAX) 'H' (PROG 10. 200.) ;

VSOUR   = 4.E6 * (EXP (-1. * (((T_INI - 1000.) / 700.) ** 2))) ;

TAB1 . 'PROCEDURE_PERSO2' = VRAI ;
TAB1 . 'MY_DATA'          = TABL ;
TAB1 . 'MY_DATA' . 'MAIL' = MAIL1 ;
TAB1 . 'MY_DATA' . 'MODE' = MOD1 ;

```

Main program

EXERCISE 3: HEAT GENERATION DEPENDING ON TEMPERATURE

Solution (bis)

- suppress loading CHA1 (heat source)
- use the **CHARTHER** procedure (called at each step iteration)
- recalculate the source (2nd member) as a function of temperatures at the beginning of the time step

```

DEBP CHARTHER T1*'TABLE'  TPS1*'FLOTTANT' ;
MA1 = T1 . 'MY_DATA' . 'MAIL' ;
MO1 = T1 . 'MY_DATA' . 'MODE' ;
* reduction du champ de températures au maillage de la source
CHT1 = T1 . 'ESTIMATION' . 'TEMPERATURES' ;
CHT2 = REDU CHT1 MA1 ;
* calcul du champ de source à partir du champ de température
CHS = 4.E6 * (EXP (-1. * ((CHT2 - 1000.) / 700.) ** 2)) ;
* création du second membre
CHFL1 = SOUR MO1 CHS ;
* sortie du second membre
T2 = TABL ;
T2 . 'ADDI_SECOND' = CHFL1 ;
FINP T2 ;

```

CHARTHER procedure

```

LT      = PROG -1.E6  1.E6 ;
LL      = 200. + (0.3 * LT) ;
LAMBDA = EVOL 'MANU' 'T' LT 'K' LL ;

HCONV   = EVOL 'MANU' 'TEMP' (PROG 0. TPSMAX) 'H' (PROG 10. 200.) ;

***TAB1 . 'CHARGEMENT'          = CHA2 ;
TAB1 . 'PROCEDURE_CHARTHER'    = VRAI ;
TAB1 . 'MY_DATA'               = TABL ;
TAB1 . 'MY_DATA' . 'MAIL'      = MAIL1 ;
TAB1 . 'MY_DATA' . 'MODE'      = MOD1 ;

```

Main program

EXERCISE 3: HEAT GENERATION DEPENDING ON TEMPERATURE

Solution (ter)

- idem
- use the **CHARTHER** procedure (called at each step iteration)
- recalculate the source (2nd member) as a function of temperatures at previous iteration (i.e. at the end of the time step)

```

DEBP CHARTHER T1*'TABLE'  TPS1*'FLOTTANT' ;
MA1 = T1 . 'MY_DATA' . 'MAIL' ;
MO1 = T1 . 'MY_DATA' . 'MODE' ;
* reduction du champ de températures au maillage de la source
CHT1 = T1 . 'WTABLE' . 'THER_COURANT' ;
CHT2 = REDU CHT1 MA1 ;
* calcul du champ de source à partir du champ de température
CHS = 4.E6 * (EXP (-1. * ((CHT2 - 1000.) / 700.) ** 2)) ;
* création du second membre
CHFL1 = SOUR MO1 CHS ;
* sortie du second membre
T2 = TABL ;
T2 . 'ADDI_SECOND' = CHFL1 ;
FINP T2 ;

```

CHARTHER procedure

```

LT      = PROG -1.E6  1.E6 ;
LL      = 200. + (0.3 * LT) ;
LAMBDA = EVOL 'MANU' 'T' LT 'K' LL ;

HCONV   = EVOL 'MANU' 'TEMP' (PROG 0. TPSMAX) 'H' (PROG 10. 200.) ;

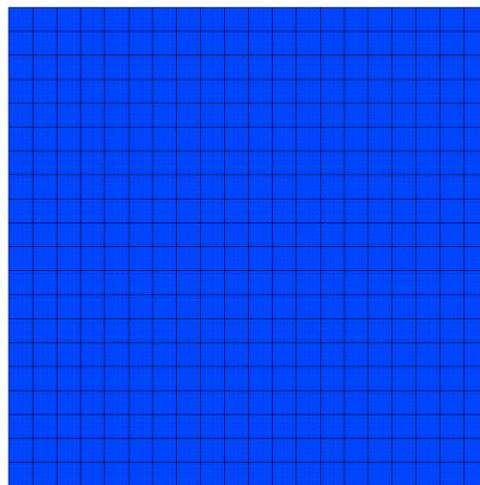
***TAB1 . 'CHARGEMENT'           = CHA1 ;
TAB1 . 'PROCEDURE_CHARTHER'     = VRAI ;
TAB1 . 'MY_DATA'                = TABL ;
TAB1 . 'MY_DATA' . 'MAIL'       = MAIL1 ;
TAB1 . 'MY_DATA' . 'MODE'       = MOD1 ;

```

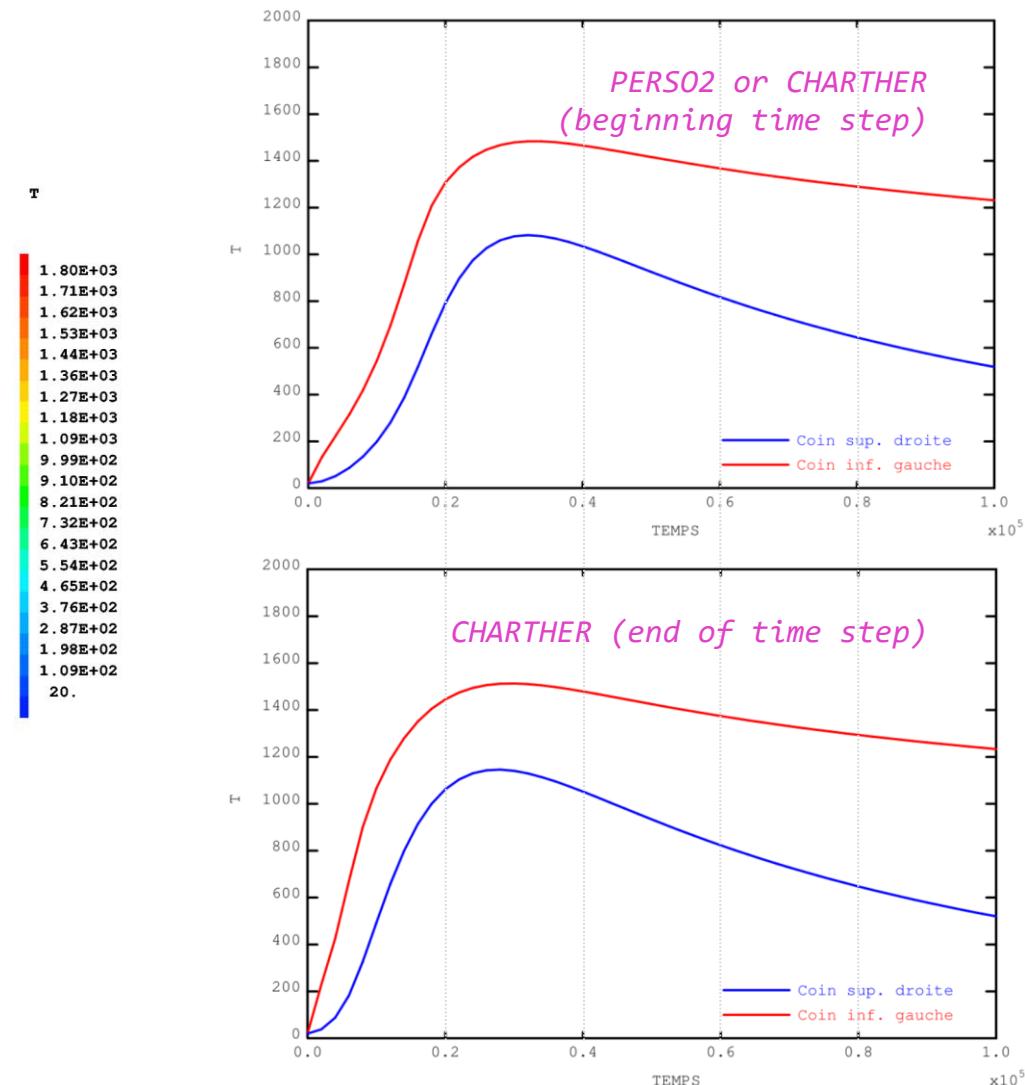
Main program

EXERCISE 3: HEAT GENERATION DEPENDING ON TEMPERATURE

■ Results



Temperatur au temps 0.00000E+00 (0%)



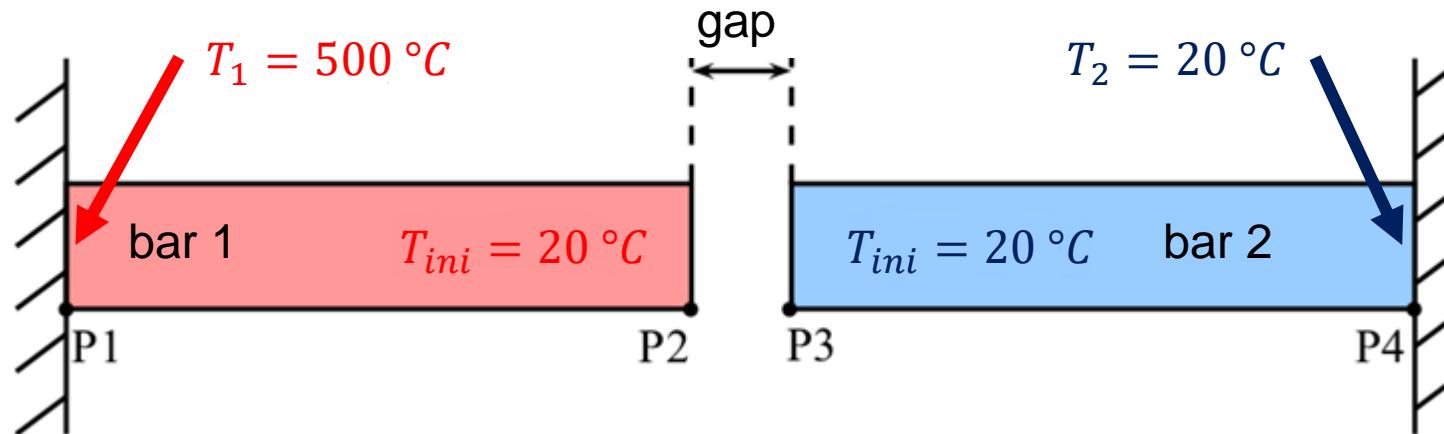
EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

DOWNLOAD THE STARTING FILE ON THE WEBSITE:

HTTP://WWW-CAST3M.CEA.FR/INDEX.PHP?PAGE=EXEMPLES&EXAMPLE=FORMATION_PASAPAS_4_INITIAL

EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

- Two bars expand and get in contact
thermal expansion
unilateral mechanical contact



- Purpose: add thermal contact, this is to say the heat transfer when the mechanical contact takes place.

It's up to you!

EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

A few indications ...

■ Useful objects

P2 et P3: points on the left/right of the gap

L2 et L3: lines on the left/right of the gap

■ Useful operators

COOR: points coordinates

RELA: relation between degrees of freedom

■ Modify thermal boundary conditions of **WTABLE**

EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

Solution

- check the **mechanical + thermal convergence**
- create a new constraint: relation between the temperatures
 T on line L2 = T on line L3 with **RELA** operator
- use **REEV_MEC** to modify the thermal boundary conditions
- calculate the gap at iteration time
- modify thermal boundary conditions in **WTABLE** as a function of the gap

```

DEBP REEV_MEC T1*'TABLE' N1*'ENTIER' ;
U1 = T1 . 'ESTIMATION' . 'DEPLACEMENTS' ;
WT = T1 . 'WTABLE' ;
TAM = T1 . 'MY_DATA' ;
* calcul du jeu
PT2 = TAM . 'POINT_2' ;
PT3 = TAM . 'POINT_3' ;
X2 = (COOR 1 PT2) + (EXTR U1 'UX' PT2) ;
X3 = (COOR 1 PT3) + (EXTR U1 'UX' PT3) ;
J1 = X3 - X2 ;
* si jeu ferme, on utilise le blocage thermique initial + le RELA
SI (J1 < EG 1.E-15) ;
    WT . 'BLOCAGES_THERMIQUES' = (TAM . 'BLOQ_0') ET
                                (TAM . 'BLOQ_1') ;
* si jeu ouvert, on utilise le blocage thermique initial seul
SINON ;
    WT . 'BLOCAGES_THERMIQUES' = TAM . 'BLOQ_0' ;
FINSI ;
FINP ;

```

REEV_MEC Procedure

```

TAB1 . 'CONVERGENCE_MEC_THE' = VRAI ;
TAB1 . 'PROCEDURE_REEV_MEC' = VRAI ;
TAB1 . 'MY_DATA' = TABL ;
TAB1 . 'MY_DATA' . 'POINT_2' = P2 ;
TAB1 . 'MY_DATA' . 'POINT_3' = P3 ;
TAB1 . 'MY_DATA' . 'BLOQ_0' = CL_TH ;
TAB1 . 'MY_DATA' . 'BLOQ_1' = RELA 'T' L2 - 'T' L3 ;

```

Main program

EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

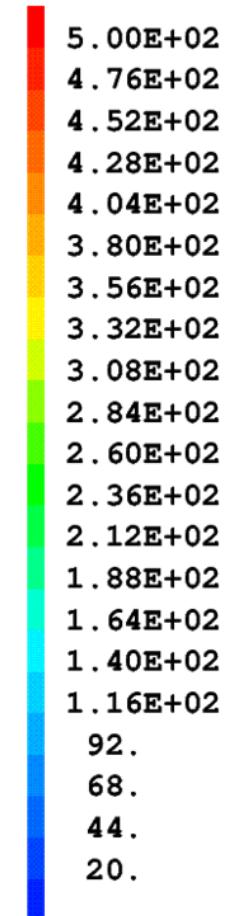
T

■ Results

Without contact: bars are insulated

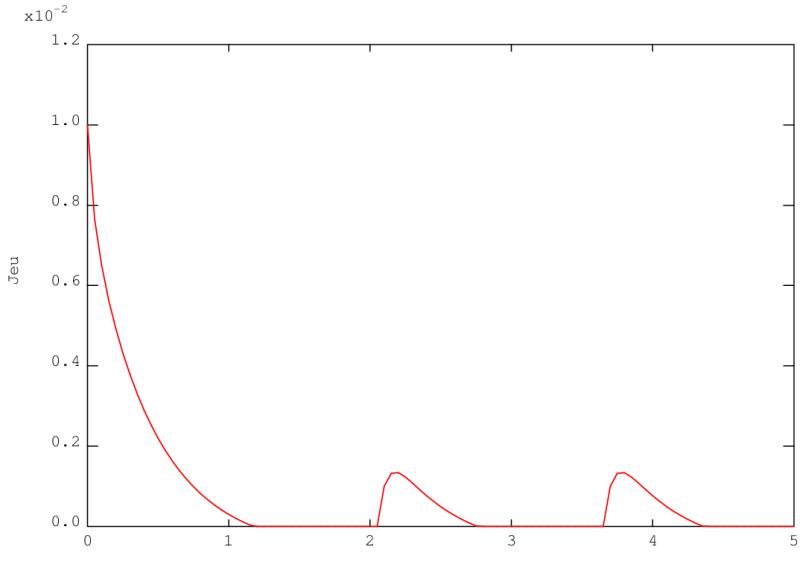


During contact: "perfect" conduction

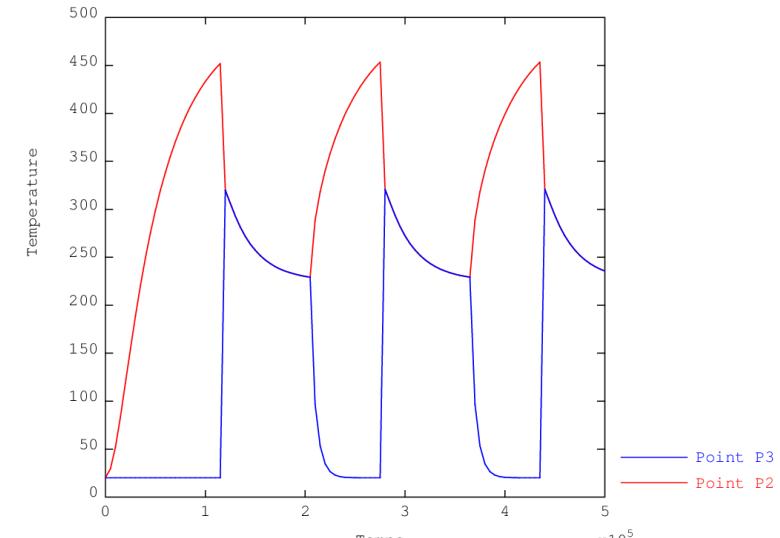


EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

■ Results



Gap distance as a function of time



*Temperatures as a function of time
on the left side and on the right side of the gap*

- Changes made in **REEV_MEC** are taken into account at the same time step
(due to thermo-mechanics convergence loop **BO_BOTH**)
- It will be different with **PERSO1**. Changes are taken into account at the next time step

EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

Solution (bis)

- convective heat transfer in a transition element
- use of an heat transfer coefficient depending on pressure
- pressure is described through a loading named 'PC'
- updating of the loading 'PC' in **REEV_MEC**

```

DEBP REEV_MEC T1*'TABLE' N1*'ENTIER' ;
R1 = T1 . 'ESTIMATION' . 'REACTIONS' ;
WT = T1 . 'WTABLE' ;
TAM = T1 . 'MY_DATA' ;
L3 = TAM . 'LIGNE_3' ;
MR = TAM . 'MAIL' ;
SI ((T1 . 'ESTIMATION' . 'TEMPS') NEG 0.) ;
* calcul de la pression de contact
PARA_P = (MAXI 'ABS' (RESU (REDU R1 L3))) / (MESU L3) ;
* chargement decrivant le parametre 'PC'
CH_PC = MANU 'CHPO' MR 'PC' PARA_P ;
EV1 = TAM . 'EVOL' ;
CHA3 = CHAR 'PC' CH_PC EV1 ;
* on ecrase le chargement global de WTABLE
WT . 'CHARGEMENT' = (TAM . 'CHAR_0') ET CHA3 ;
FINS ;
FINP ;

```

REEV_MEC procedure

```

** Modele de convection variable entre les barreaux
MRACC = RACC (1.1 * JEU_INI) L2 L3 ;
MODRACC = MODE MRACC 'THERMIQUE' 'CONVECTION' ;
HVSP = EVOL 'MANU' 'PC' (PROG 0. 5.E8 5.1E8)
          'H' (PROG 0. 1.E4 1.E4) ;
MATRACC = MATE MODRACC 'H' HVSP ;
MOD_TH = MODT1 ET MODT2 ET MODRACC ;
MAT_TH = MATT1 ET MATT2 ET MATRACC ;

** Chargement initial decrivant la pression de contact
CH_PC = MANU 'CHPO' MRACC 'PC' 0. ;
CHA2 = CHAR 'PC' CH_PC EV1 ;

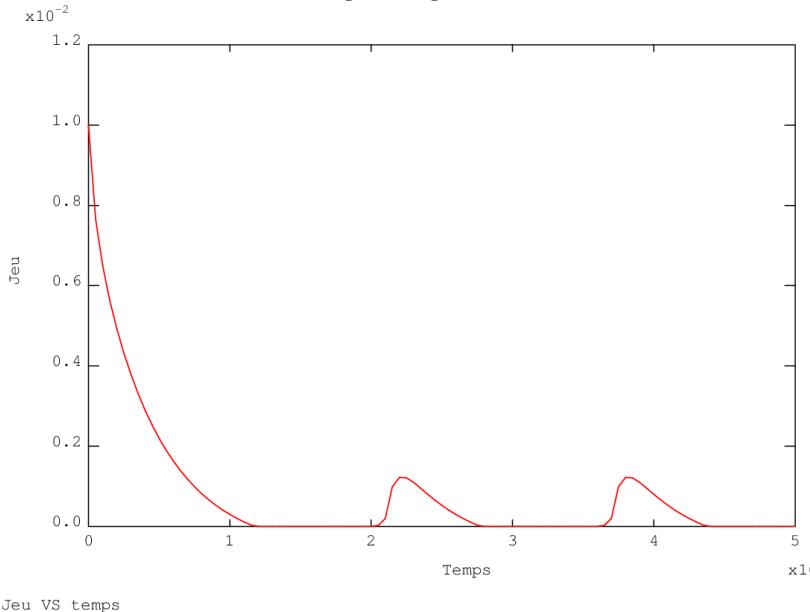
TAB1 . 'CHARGEMENT'           = CHA1 ET CHA2 ;
TAB1 . 'CONVERGENCE_MEC_THE' = VRAI ;
TAB1 . 'PROCEDURE_REEV_MEC'   = VRAI ;
TAB1 . 'MY_DATA'              = TABL ;
TAB1 . 'MY_DATA' . 'LIGNE_3' = L3 ;
TAB1 . 'MY_DATA' . 'MAIL'    = MRACC ;
TAB1 . 'MY_DATA' . 'EVOL'    = EV1 ;
TAB1 . 'MY_DATA' . 'CHAR_0'  = CHA1 ;

```

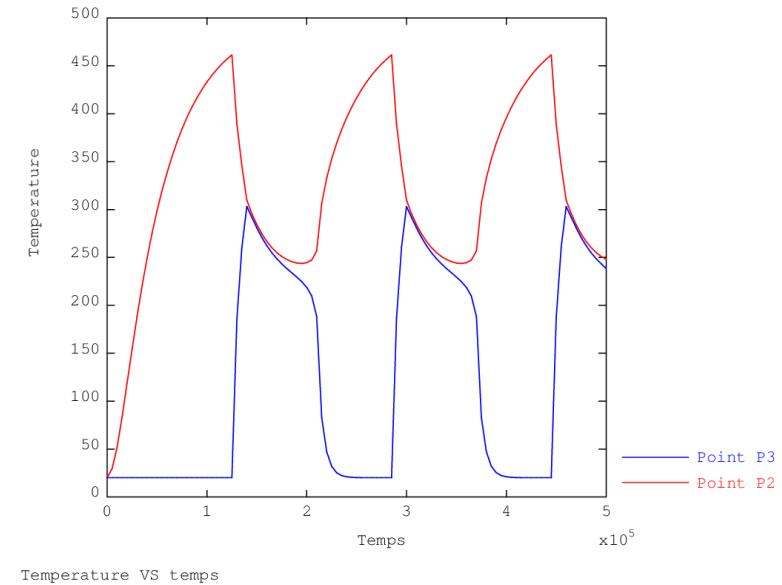
Main program

EXERCISE 4: THERMO-MECHANICAL GAP CLOSING

■ Results (bis)



Gap distance as a function of time



*Temperatures as a function of time
on the left side and on the right side of the gap*

- The heat transfer is made with different temperatures over the gap
- The temperature evolution is smoother

ANNEXES

ANNEX: ALGORITHMIC CONTROL PARAMETERS

■ General

NB_BOTH (ENTIER)

Maximum number of iterations for the thermo-mechanical convergence loop

MAXITERATION (ENTIER)

Maximum number of iterations (49)

■ Mechanics

PRECISION (FLOTTANT)

Criterion to compare the imbalance (1.E-4)

FTOL (FLOTTANT)

Tolerance for strength equilibrium

MTOL (FLOTTANT)

Tolerance for moments equilibrium

GRANDS_DEPLACEMENTS (LOGIQUE)

Reference configuration = deformed configuration

PREDICTEUR = 'HPP'

*Small displacements hypothesis → 1st convergence
then use the*

large displacements hypothesis → 2nd convergence

Precision for the constitutive laws integration local problem (1.E-8)

Forced convergence use or not in case of non convergence (VRAI)

Max. number of sub-steps during forced convergence (200)

Number of steps over which non convergence is tested

PRECISINTER (FLOTTANT)

CONVERGENCE_FORCEE (LOGIQUE)

MAXSOUSPAS (ENTIER)

DELTAITER (ENTIER)

ANNEX: ALGORITHMIC CONTROL PARAMETERS

■ Thermal

PROCEDURE_THERMIQUE (MOT)

Procedures to call:

- **NONLINEAR** *TRANSNON procedure*
- **LINEAR** *TRANSLIN procedure*
- **DUPONT** *DUPONT2 procedure*

Relaxation coefficient for the θ -method (1)

RELAXATION_THETA (FLOTTANT)

■ Thermo-mechanical coupling

CONVERGENCE_MEC_THE (LOGIQUE)

*Indicates that the thermo-mechanical loop should be repeated in case of dependence (**FAUX**)*

CRITERE_COHERENCE (FLOTTANT)

*Precision for the thermo-mechanical convergence, tested on thermal results (**1.E-2**)*

PROJECTION (LOGIQUE)

*Indicates that the thermal and mechanical meshes are different while the problem is coupled (**FAUX**)*

ANNEX: IMBALANCE MINIMIZATION IN GIBIANE

■ Simplified UNPAS algorithm (see the **@SOLVMEC** procedure) [small displacements / strains]

```
fex1 = TIRE char tps1 'MECA' ;
f1x1 = TIRE char tps1 'DIMP' ;
resi0 = fex1 + fr0 - fi0 + f1x1 - f1x0 ;
fref = MAXI 'ABS' (fex1 + fr0) ;
k = (RIGI mod mat) ET blo ;
REPE b1 100 ;
  ddu = RESO k resi0 ;
  u1 = u0 + ddu ;
  fr1 = REAC blo u1 ;
  eps1 = EPSI mod u1 ;
  sig1 = ... COMP eps0 eps1 ...
  fi1 = BSIG mod sig1 ;
  resi1 = fex1 + fr1 - fi1 ;
  crit1 = (MAXI 'ABS' resi1) / fref ;
  SI (crit1 < precis) ;
    QUIT b1 ;
  FINSI ;
  u0 = u1 ;
  resi0 = resi1 ;
FIN b1 ;
```

ANNEX: IMBALANCE MINIMIZATION IN GIBIANE

■ Simplified UNPAS algorithm (see the @SOLVMEC procedure) [large displacements, updated lagrangian]

```

fex1 = TIRE char tps1 'MECA' ;
flx1 = TIRE char tps1 'DIMP' ;
resi0 = fex1 + fr0 - fi0 + flx1 - flx0 ;
fref = MAXI 'ABS' (fex1 + fr0) ;
conf0 = FORM u0 ;
u00 = u0 ;
k = (RIGI mod mat) ET blo ET (KSIG mod sig0) ;
REPE b1 100 ;
  ddu = RESO k resi0 ;
  u1 = u0 + ddu ;
  fr1 = REAC blo u1 ;
  du = u1 - u00 ;
  deps = EPSI mod du ;
  eps1 = eps0 + deps ;
  sig1 = ... (COMP eps0 eps1, EXCO, ...) ;
  sig1 = PICA mod sig1 du ;
  FORM u1 ;
  fi1 = BSIG mod sig1 ;
  FORM conf0 ;
  resi1 = fex1 + fr1 - fi1 ;
  crit1 = (MAXI 'ABS' resi1) / fref ;
  SI (crit1 < precis) ;
  QUIT b1 ;
  FINSI ;
  u0 = u1 ;
  resi0 = resi1 ;
FIN b1 ;

```

moving to the current configuration (beginning of time step)

strain increment

moving stresses to the final configuration

integration on the final configuration

ANNEX: CONVERGENCE CRITERIA (UNPAS)

■ Convergence norms (after 1st RESO)

$$F^{ref} = \frac{\max_{\text{dof deppl}} |\delta\{U\}_1^1 \cdot \{F\}_1^{S+V} - \{\lambda\}_1^1 \cdot (\{d\}_1 - [A]^T \cdot \{U\}_1^0)|}{\max_{\text{dof deppl}} |\delta\{U\}_1^1| + \text{xsmall}} + \frac{\max_{\text{dof deppl}} |\delta\{U\}_1^1| \cdot \max|[B]\{\sigma\}_1^0|}{\max_{\text{dof deppl}} |\delta\{U\}_1^1| + \text{xsmall}} + \max|[B]\{\sigma\}_1^0|$$

$$M^{ref} = \frac{\max_{\text{dof rota}} |\delta\{U\}_1^1 \cdot \{F\}_1^{S+V} - \{\lambda\}_1^1 \cdot (\{d\}_1 - [A]^T \cdot \{U\}_1^0)|}{\max_{\text{dof rota}} |\delta\{U\}_1^1| + \text{xsmall}} + \frac{\max_{\text{dof deppl}} |\delta\{U\}_1^1| \cdot \max|[B]\{\sigma\}_1^0|}{\max_{\text{dof rota}} |\delta\{U\}_1^1| + \text{xsmall}} + \text{xsmall}$$

■ Unbalance measure (at each UNPAS iteration)

$$\mathbf{xconv} = \frac{\max \left\{ \max_{\text{ddl deppl}} |\{F\}_1^{S+V} - [A]^T \cdot \{\lambda\}_1^{i+1} - [B]\{\sigma\}_1^{i+1}| ; \max |[A]^T \cdot (\{\lambda\}_1^{i+1} - \{\lambda\}_1^i)| \right\}}{F^{ref}}$$

$$\mathbf{xconvm} = \frac{\max_{\text{ddl rota}} |\{F\}_1^{S+V} - [A]^T \cdot \{\lambda\}_1^{i+1} - [B]\{\sigma\}_1^{i+1}|}{M^{ref}}$$

ANNEX: CONVERGENCE CRITERIA (UNPAS)

■ Measure of strain increment variation between 2 iterations

$$\mathbf{DEPSTDm} = \max \left| \Delta\{\varepsilon\}_1^{i+1} - \Delta\{\varepsilon\}_1^i \right|$$

■ Convergence is reached if

$$\begin{aligned}\mathbf{XCONV} &< \text{PRECISION} \\ \mathbf{XCONVM} &< \text{PRECISION} \\ \mathbf{DEPSTDm} &< \text{PRECISION}\end{aligned}$$

ANNEX: FORCED CONVERGENCE TO BE FINISHED...

■ If non convergence is confirmed, the step is re-calculated:

- state at the beginning of the step is redefined
- the last “out of equilibrium” state with minimal imbalance is chosen
- at the beginning of time step, are overwritten:
 - the instant
 - the material properties
 - the displacements
 - the internal variables
 - the inelastic strains
 - the stresses
- UNPAS is re-started with a time-step equal to zero
- no load increment, only disequilibrium
- allows to find a solution with the right behavior and equilibrium but with a different loading path


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